3.1 Modified Cam Clay Model with Drained Condition

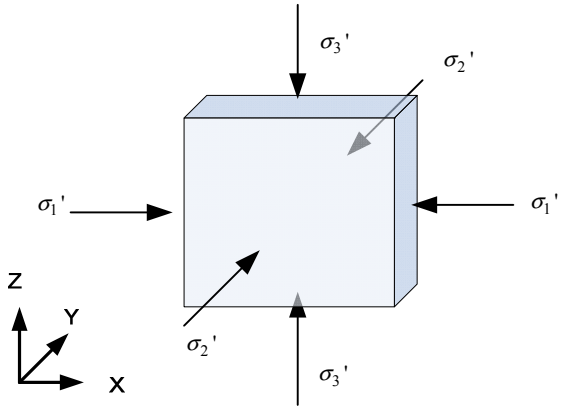
REFERENCE	Sam Helwany[1]
ELEMENTS	Solid elements
MODEL FILENAME	Material01.gts

Nonlinear tri-axial tension and compression are carried out using modified Cam-clay model with drained condition. The objective of this example is to verify the material model in reproducing the theoretical effective pressure-shear stress response. The theoretical response can be obtained analytically from the following set of equations:

$$\sigma_1' = \sigma_2' \neq \sigma_3'$$
$$p' = -\frac{\sigma_1' + \sigma_1' + \sigma_3'}{3}$$
$$q = |\sigma_3' - \sigma_1'|$$
$$\frac{\Delta q}{\Delta p'} = 3$$

Tri-axial tension and compression tests are performed for normally consolidated and over-consolidated soil specimen as depicted in Figure 3.1.1. Initial stage is set up where pressure is applied along the x-, y- and z-directions to reproduce the in-situ stresses. In the subsequent stage, additional pressure is applied in the z-direction for loading test. For the unloading case, specified displacement load is applied in the z-direction.

Figure 3.1.1
Drained tri-axial test



Material data	Poisson's ratio	$\nu = 0.3$
	OCR	*1.0 : normal consolidation*
		2.0 : over consolidation
	κ	*0.026*
	λ	*0.174*
	e_0	*0.889*
	M	*1.0*
	Drained	

Figure 3.1.2
Mean effective pressure vs. shear stress for loading/unloading in normal consolidation condition

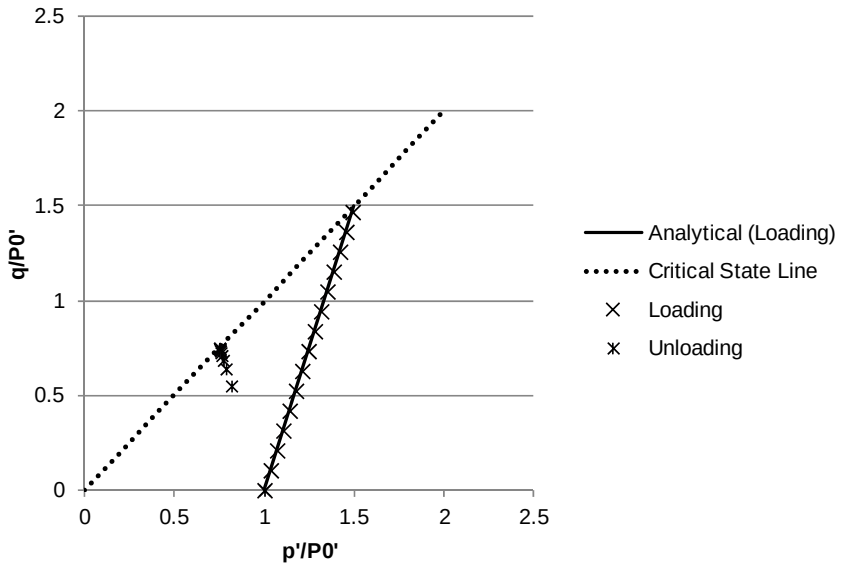


Figure 3.1.3
Axial strain vs. shear stress for loading/unloading in normal consolidation condition

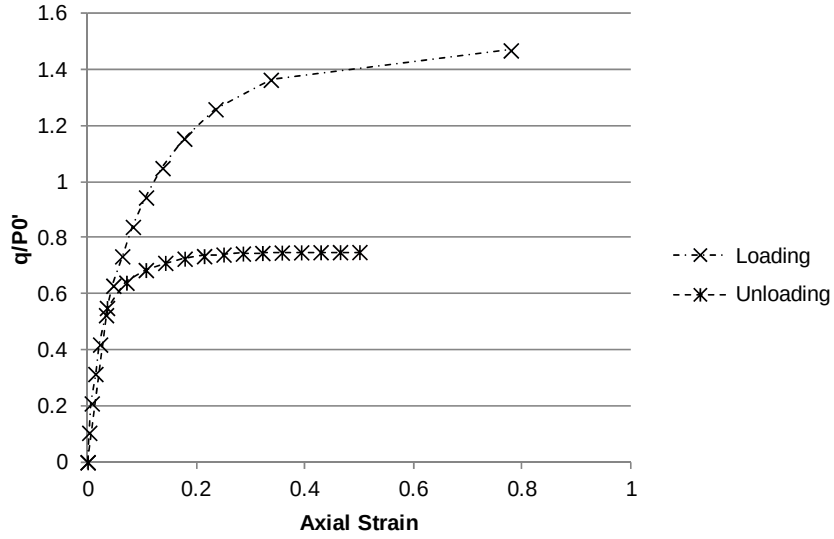


Figure 3.1.4
Mean effective pressure vs. shear stress for loading/unloading in over-consolidation condition (OCR=2)

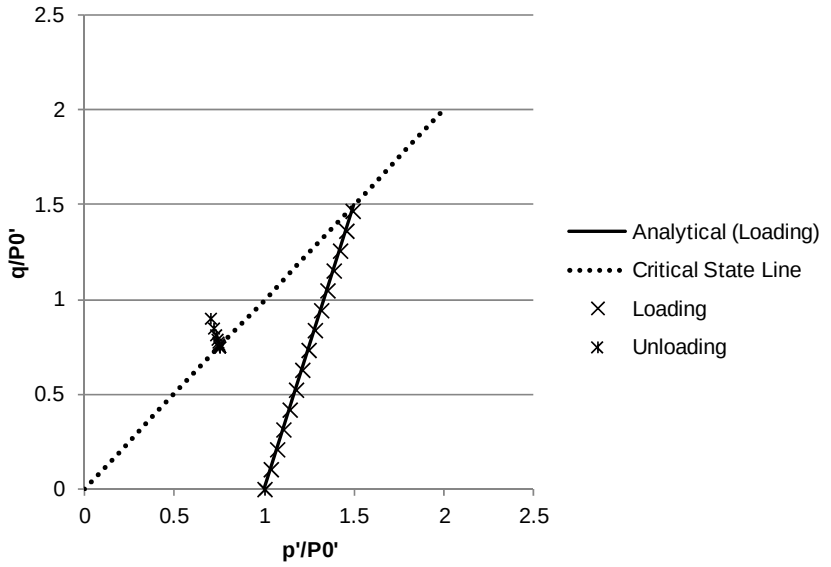
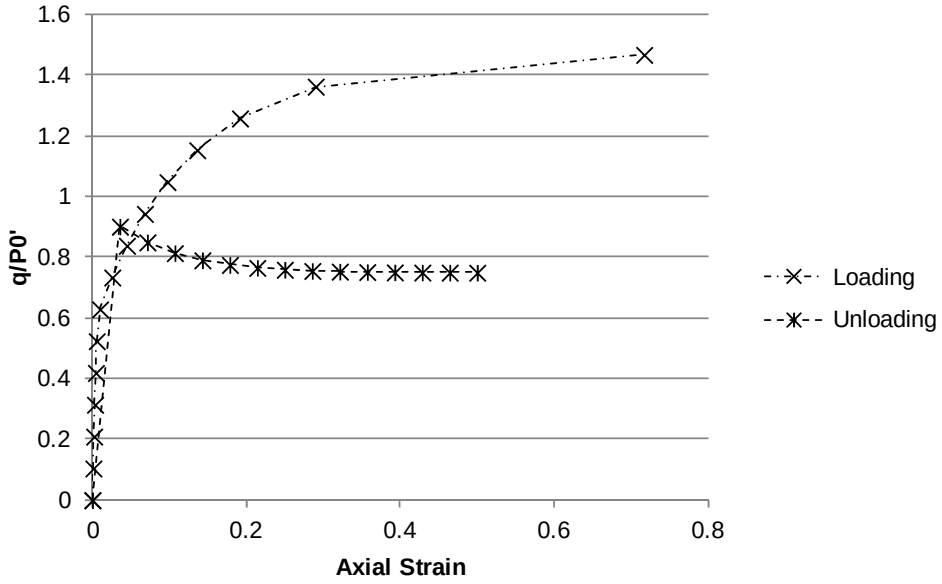


Figure 3.1.5
Axial strain vs. shear stress for loading/unloading in over-consolidation condition (OCR=2)



3.2 Modified Cam Clay Model with Undrained Condition

REFERENCE	Wood, D. M.[2]
ELEMENTS	Solid elements
MODEL FILENAME	Material02-A.gts, Material02-B.gts

Nonlinear tri-axial compression tests are carried out using modified Cam-clay model with undrained condition. The objective of this example is to verify the material model in reproducing the theoretical effective pressure-shear stress response in undrained condition. The theoretical response can be obtained analytically from the following set of equations:

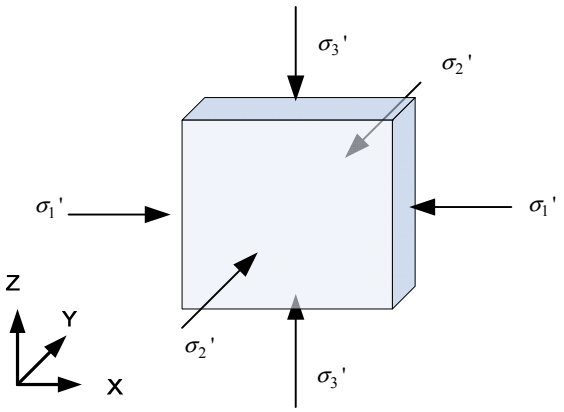
$$\frac{p_i'}{p'} = \left(\frac{M^2 + \eta^2}{M^2 + \eta_i^2} \right)^{\Lambda}$$

$$\Lambda = \frac{\lambda - \kappa}{\lambda}$$

$$\eta = \frac{q}{p'}$$

p_i , η_i define the effective stress state at impending yield. Tri-axial compression tests are performed for over-consolidated soil specimen as depicted in Figure 3.2.1. Initial stage is set up where pressure is applied along the x-, y- and z-directions to reproduce the in-situ stresses. In the subsequent stage, additional displacement load is applied in the z-direction for loading test.

Figure 3.2.1
Undrained tri-axial test



Material data	Poisson's ratio	$\nu = 0.3$
	OCR	*1.5*
		2.5
	κ	*0.026*
	λ	*0.174*
	e_0	*0.889*
	M	*1.0*
	Undrained	

Figure 3.2.2
Mean pressure vs. shear stress for loading in over consolidation condition (OCR=1.5)

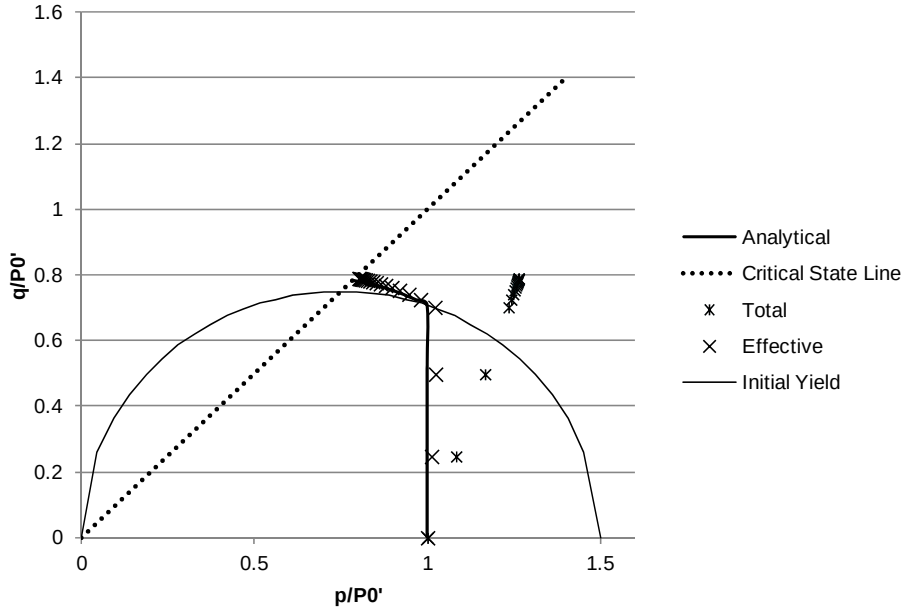
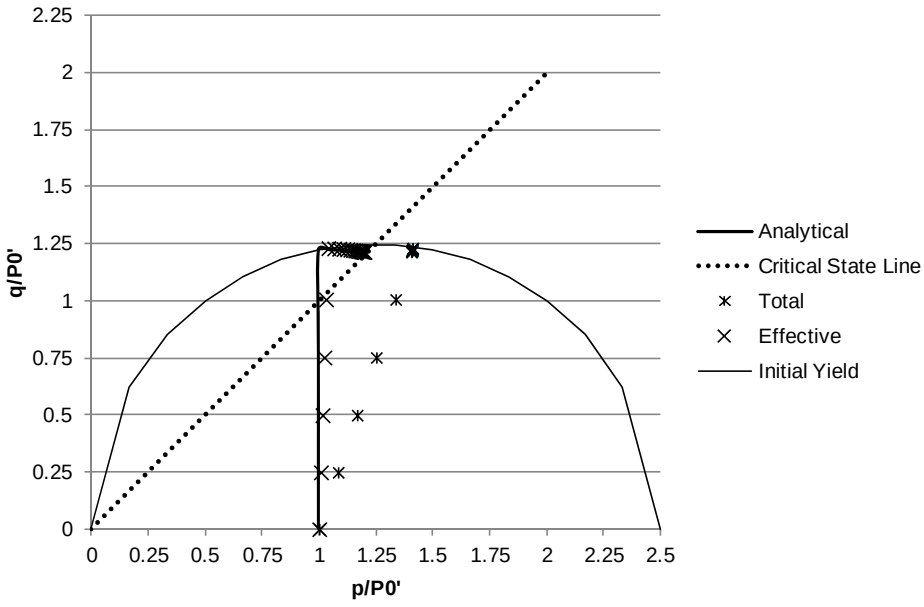


Figure 3.2.3
Mean pressure vs. shear stress for loading in over consolidation condition (OCR=2.5)



3.3 Internally Pressurized Spherical Shell Composed of von-Mises Material

REFERENCE	Hill, R. [3]
ELEMENTS	Solid elements, axisymmetric elements
MODEL FILENAME	Material03_1.gts, Material03_2.gts

A spherical shell model is depicted in Figure 3.3.1. The model is composed of the elastic-perfectly plastic von Mises material model. The model is subjected to a uniform internally pressure P. Nonlinear analysis is carried out with arc-length method to obtain the pressure-deflection response. Solutions are obtained using both solid and axisymmetric elements. The inner and outer radii (a and b) are 1m and 2m, respectively.

Analytical solution derived by Hill is adopted for comparison. The closed form solution for the pressure-deflection relations can be represented as:

$$\begin{cases} u_b = \dfrac{3Pb}{2E(b^3/a^3-1)}(1-\upsilon) & (P < P_0) \\ u_b = \dfrac{\sigma_{y_0} c^3}{Eb^2}(1-\upsilon) & (P \geq P_0) \end{cases}$$

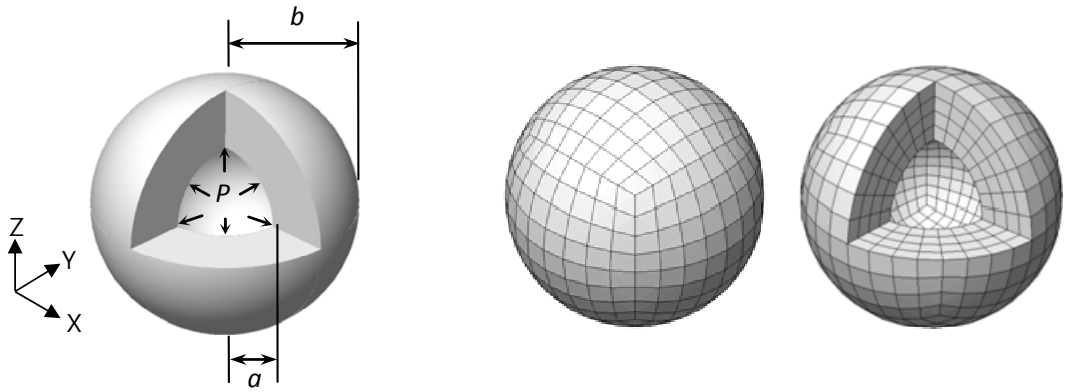
where P_0 is the pressure at the onset of plasticity and plastic radius, c can be obtained from:

$$P = 2\sigma_{y_0} \ln\left(\frac{c}{a}\right) + \frac{2\sigma_{y_0}}{3}\left(1-\frac{c^3}{b^3}\right)$$

The limiting load is reached when $c = b$, and can be represented as:

$$P_{\lim} = 2\sigma_{y_0} \ln\left(\frac{b}{a}\right)$$

Figure 3.3.1
Geometry and mesh of spherical shell model



Material data	Young's modulus	*E = 21.0 GPa*
	Poisson's ratio	*ν = 0.3*
	Yield criteria	*von-Mises*
	Yield stress	*240 MPa*

Figure 3.3.2
Graph of internal pressure vs. radial displacement at outer face

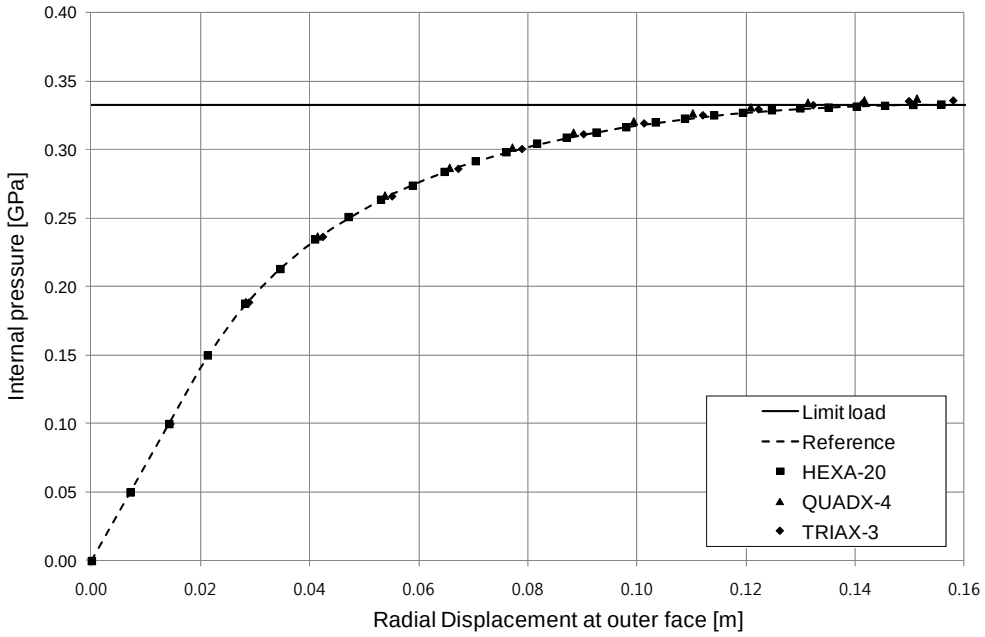


Table 3.3.1 Limit pressure obtained using solid and axisymmetric elements

	P_{inter} [GPa]
Reference	*0.3327*
HEXA-20	*0.3200*
QUADX-4	*0.3396*
QUADX-8	*0.3396*
TRIAX-3	*0.3368*
TRIAX-6	*0.3328*

3.4 A V-notch Bar composed of Elastic-Perfectly Plastic Tresca Material Model

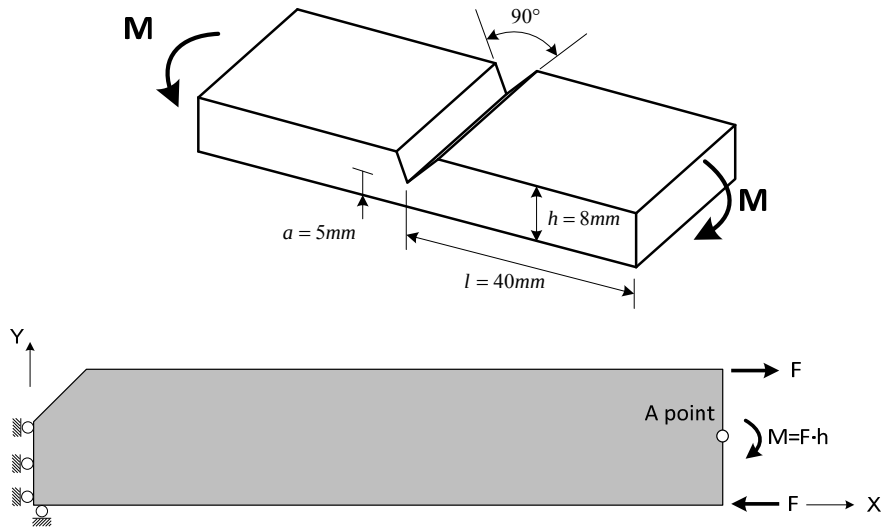
REFERENCE	Green, A.P.[4], Chakrabarty, J.[5]
ELEMENTS	Plane strain elements
MODEL FILENAME	Material04.gts

A wide rectangular metal bar containing a deep 90° V-shaped notch subjected to pure bending is considered. Figure 3.4.1, shows the geometry of the model. The model is considered as a plain strain problem. Elastic-perfectly plastic Tresca material behavior is assumed. The bending moment is applied by means of two opposite nodal forces as indicated in Figure 3.4.1. 8-node quadrilateral plane strain elements are used for the nonlinear analysis. Utilizing the symmetry of the problem, only half of the model is analyzed with symmetric boundary conditions enforced. The obtained bending moment is compared with an analytic solution. The analytic solution for the upper bound of the moment per unit width is available based on the theory of slip line field:

$$M_u \approx 0.623ca^2$$

where a is the thickness of the bar in the neck and c is the shear strength, which for the Tresca material model is half of the yield stress, $c = \sigma_Y / 2$

Figure 3.4.1
V-shaped notch problem



Material data	Young's modulus	*E = 210000 MPa*
	Poisson's ratio	*ν = 0.3*
	Yield criteria	*Tresca*
	Yield Stress	*240 MPa*

Figure 3.4.2
Normalized bending moment obtained using plane strain elements

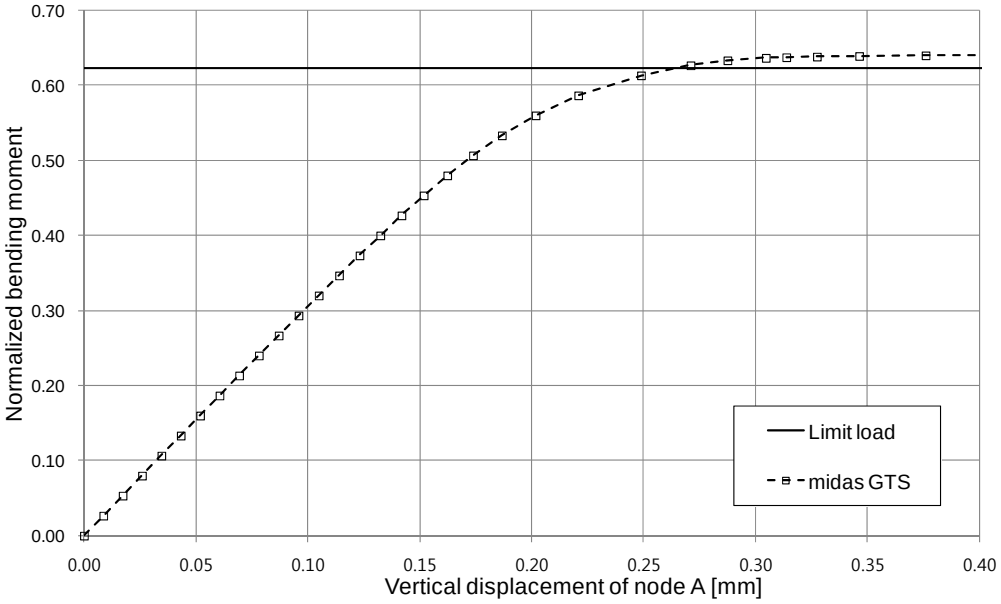


Table 3.4.1 Normalized bending moment

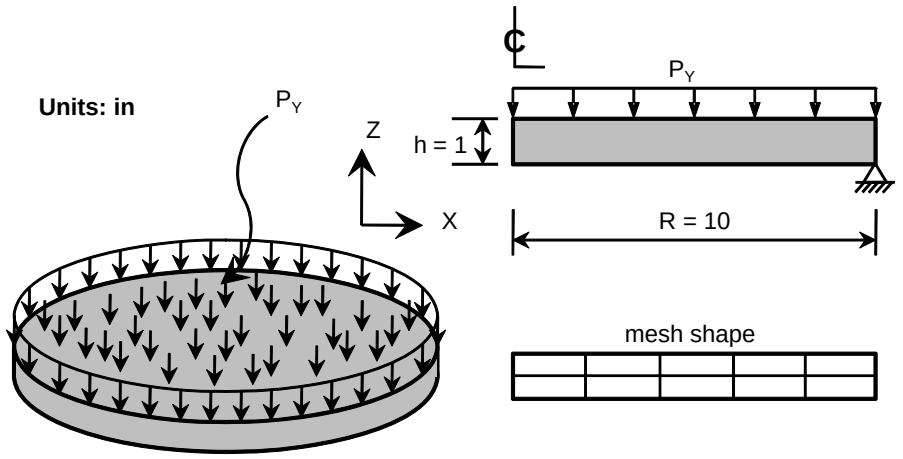
	Normalized bending moment
Reference	*0.623*
QUAD-8	*0.641*

3.5 Uniformly Loaded Circular Plate

REFERENCE	Owen et al.[6]
ELEMENTS	Axisymmetric elements
MODEL FILENAME	Material05.gts

Figure 3.5.1 shows a circular plate model simply supported around its outer rim under uniform pressure. The circular plate exhibits material nonlinear behavior. The constitutive model follows the elastic-perfectly plastic von Mises model. The limit pressure is determined using various types of axisymmetric elements.

Figure 3.5.1
Circular plate model



Material data	Young's modulus	*$E = 1.0 \times 10^7$ psi*
	Poisson's ratio	*$\nu = 0.24$*
	Model type	*Von-Mises*
	Yield stress	*16000.0 psi*
Load data	Nodal Force	*264 lbf/in*

Figure 3.5.2
Pressure vs. center deflection of circular plate model obtained using axisymmetric elements

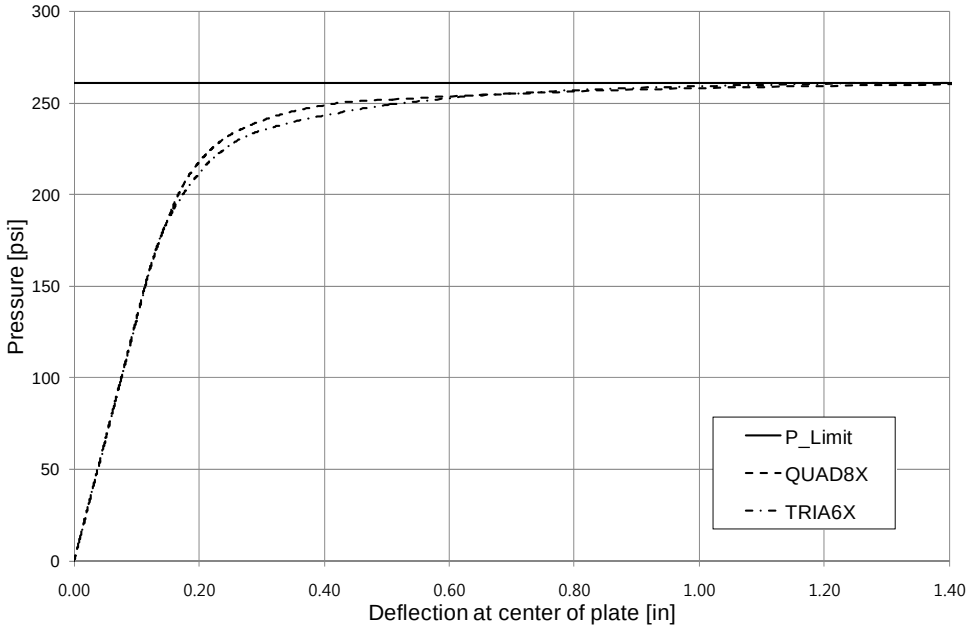


Table 3.5.1 Limit pressure

	P_{lim} [psi]
Reference	*260.8*
QUAD-8	*261.2*
TRIA-6	*261.4*

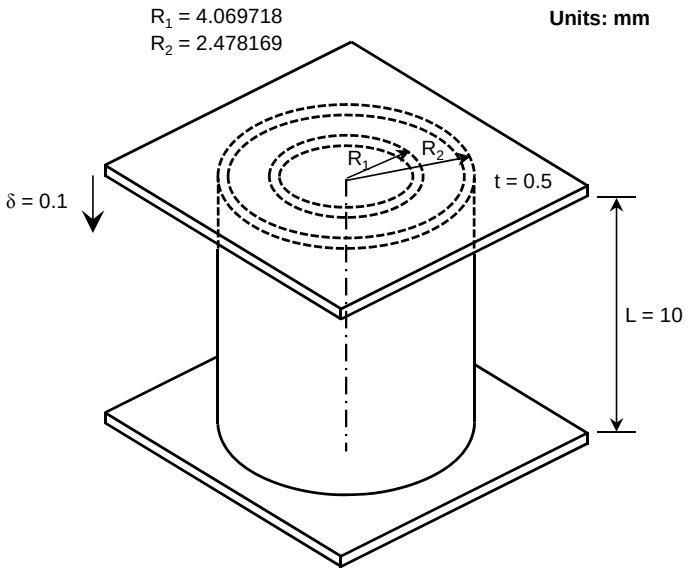
* $P_{lim} \approx \frac{6.52 M_y}{R^2}, \quad M_y \equiv \frac{\sigma_y h^2}{4}$

3.6 Two Coaxial Tubes

REFERENCE	Crandall et al.[7]
ELEMENTS	Solid elements
MODEL FILENAME	Material06.gts

Figure 3.6.1 shows two coaxial tubes made out of steel (1020-CR) and aluminum alloy (2024-T4) compressed between two rigid flat ends. Both tubes are assumed to follow the von Mises elastic-perfectly plastic material model with the associated flow rule.

Figure 3.6.1
Two coaxial tubes



Material data (Inner tube: steel 1020-RC)	Young's modulus	$E_{steel} = 1.84375 \times 10^5$ *N/mm²*
	Poisson's ratio	$\nu_{steel} = 0.3$
	Model type	*von-Mises*
	Yield stress	*590 N/mm²*
Material data (Outer tube: aluminum alloy 2024-T4)	Young's modulus	$E_{alloy} = 7.6 \times 10^4$ *N/m²*
	Poisson's ratio	$\nu_{alloy} = 0.3$
	Model type	*von-Mises*
	Yield stress	*380 N/mm²*
Load data	Prescribed displacement	*0.1mm*

Figure 3.6.2
Reaction force vs. applied displacement curve obtained using hexahedron elements

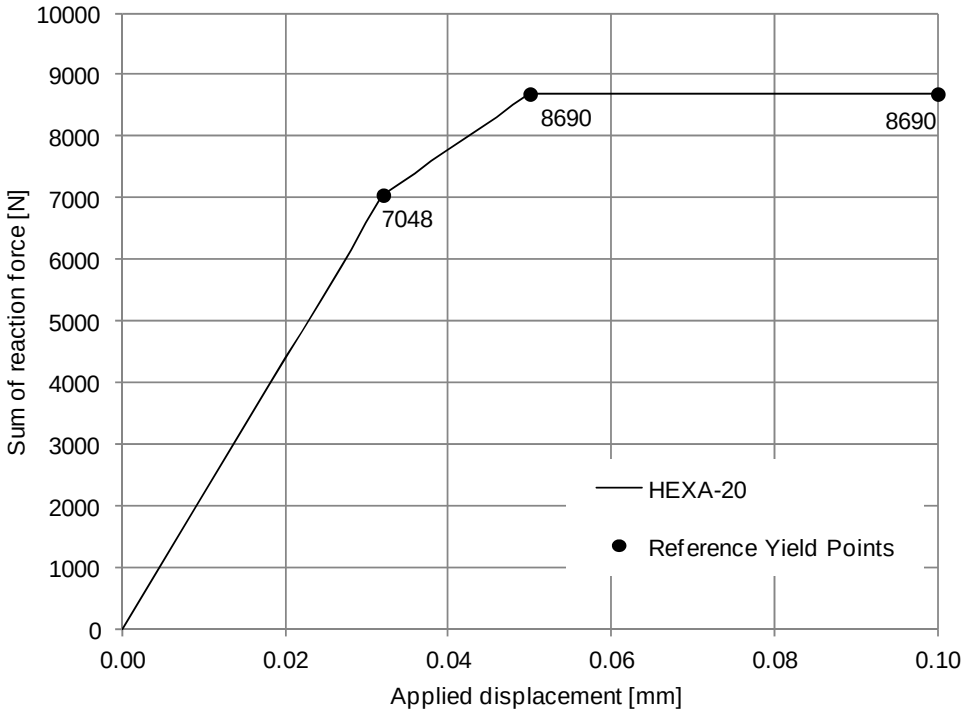


Table 3.6.1 Reaction force at the bottom surface with the entire model at plasticity state

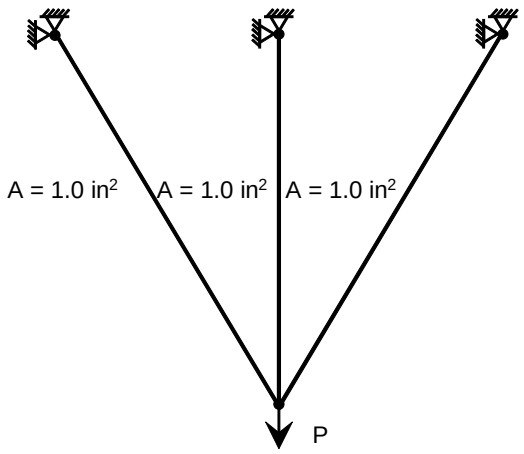
	$F_R[N]$
Reference	*8690.0*
HEXA-8	*8681.1*
PENTA-6	*8673.3*
HEXA-20	*8681.1*
PENTA-15	*8683.7*

3.7 Residual Stress Problem

REFERENCE	Crandall et al.[7]
ELEMENTS	Truss elements
MODEL FILENAME	Material07.gts

Figure 3.7.1 shows a chain hoist attached to the ceiling through three tie trusses, which are made out of cold-rolled steel with the yield strength of σ_y . The three-truss structure is loaded in the vertical direction and then unloaded. While the structure is loaded, the center truss member yields first while the side members remain elastic. When the frame is fully loaded, all the three trusses become fully plastic.

Figure 3.7.1
Three member truss model



Material data	Young's modulus	$E = 30.0 \times 10^6\ psi$
	Poisson's ratio	$\nu = 0.3$
	Model type	*von-Mises*
	Yield stress	*30000.0 psi*
Section Property	Cross-sectional area	$A = 1.0\ in^2$

Figure 3.7.2
Stress-deflection curve
of the center member

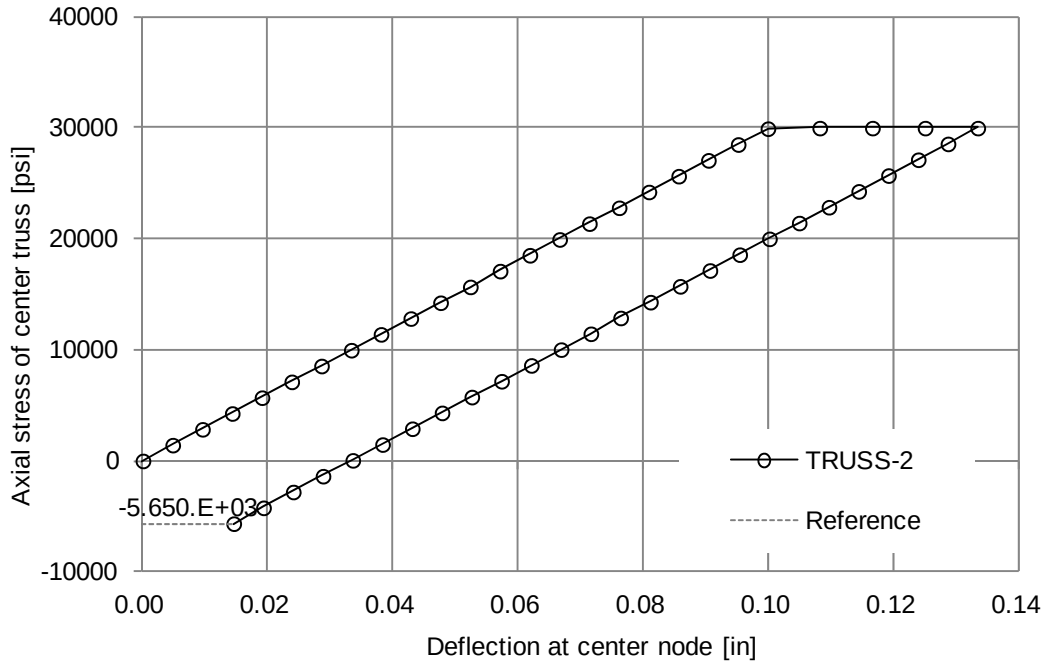


Table 3.7.1 Residual stress results of center member after loading-unloading

	$\delta_{elastic}$ [*in*]	σ_{res} [*psi*]
Reference	*0.1000*	*-5650*
TRUSS-2	*0.0998*	*-5650*

3.8 End-Loaded Tapered Cantilever Composed of Tresca Material Model

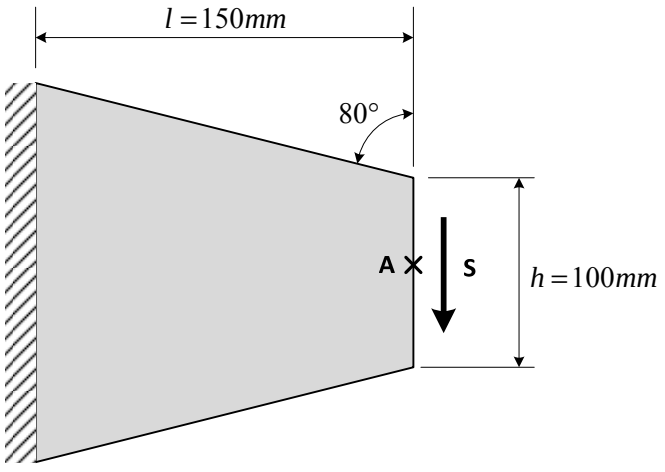
REFERENCE	Green, A.P.[8]
ELEMENTS	Plane strain elements, Tresca
MODEL FILENAME	Material08.gts

Figure 3.8.1 shows a cantilevered tapered beam subjected to uniform shear force at the free edge. Elastic-perfectly plastic Tresca material model behavior is assumed. The cantilever problem is discretized using plane strain elements to obtain the limit load. Comparison is made with an analytical solution based on the theory of slip line field. The normalized upper bound for shear pressure is expressed as:

$$\frac{S_u}{c} \approx 0.775$$

where c is the shear strength, which for the Tresca material model is half of the yield stress, $c = \sigma_Y / 2$

Figure 3.8.1
End-loaded tapered cantilever model



Material data	Young's modulus	*E = 210000 MPa*
	Poisson's ratio	*ν = 0.3*
	Yield criteria	*Tresca*
	Yield stress	*240 MPa*
Section data	Plane strain	

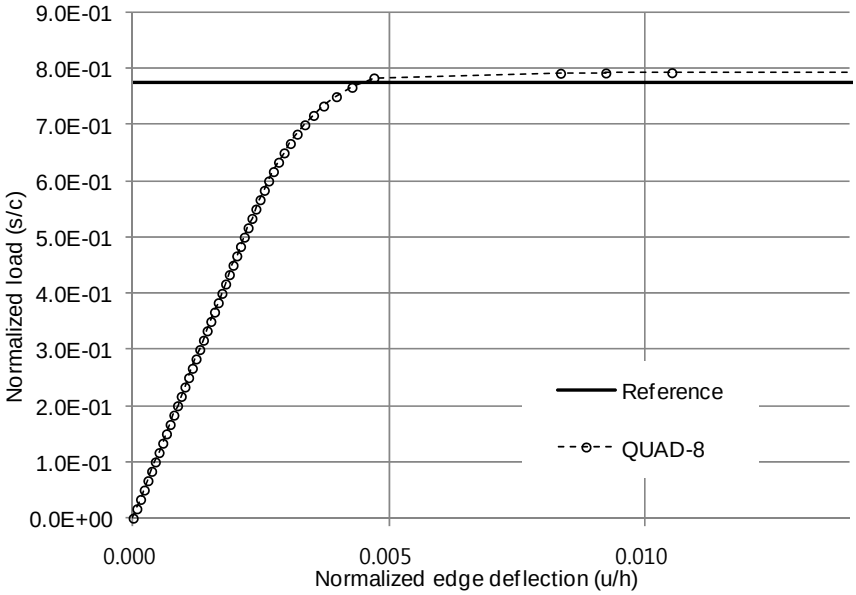


Figure 3.8.2 Normalized load vs. normalized edge deflection of tapered beam model using plane strain elements

Table 3.8.1 Normalized bending moment compared with analytical solution

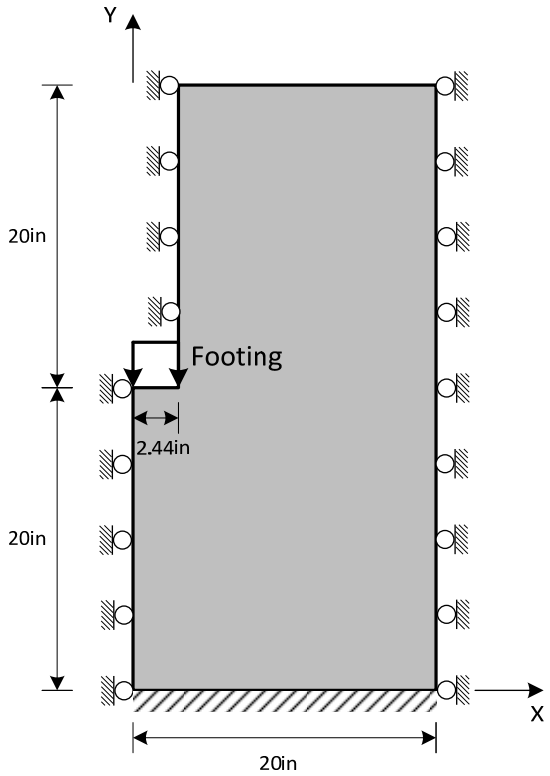
	Normalized bending moment
Reference	*0.775*
QUAD-8	*0.798*

3.9 Strip Footing in Sand

REFERENCE	Duncan, J. M. et al. [9]
ELEMENTS	Plane strain elements, Footing, Cohesion varing
MODEL FILENAME	Material09.gts

Figure 3.9.1 shows the symmetric computational model for the strip footing in sand. Duncan-Chang material behavior is assumed. Nonlinear finite element analysis is carried out where prescribed displacement is applied on the footing surface. Plane strain elements are used to obtain the pressure vs. settlement response. The results are compared with computational solution and experimental observations found in the reference.

Figure 3.9.1
Half model of strip footing problem



Material data	Model Type	*Hyperbolic (Duncan-Chang)*
	Initial loading modulus	$K = 300$
	Initial stiffness ratio	$n = 0.55$
	Failure Ratio	$R_f = 0.83$
	Cohesion	$C = 0$
	Friction angle	$\phi = 35.5°$
	Unit Weight	$91\ lb/ft^3$
	Poisson ratio	$\nu = 0.35$

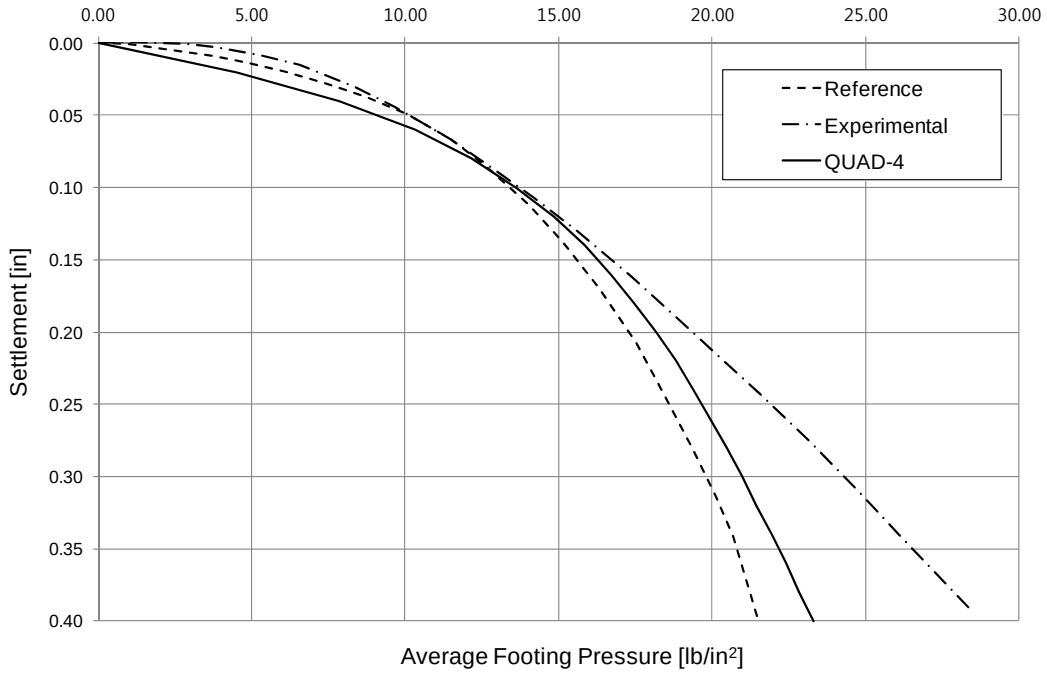


Figure 3.9.2
Settlement vs. average footing pressure obtained using plane strain elements

3.10 Bearing Capacity of Smooth Square Footing on a Frictionless Material

REFERENCE	Shield et al. [10], Chen[11]
ELEMENTS	Solid elements
MODEL FILENAME	Material10.gts

Figure 3.10.1 shows a quarter model of a square footing problem on a frictionless material. Nonlinear finite element analysis is carried out to determine the bearing capacity of the problem with symmetric boundary conditions imposed on the planes of symmetry. Smooth footing is simulated by prescribing displacements in the vertical direction while releasing the displacements in the horizontal directions. Mohr-Coulomb material model with zero friction and dilatancy angles is used to model the material behavior.

For comparison, the upper and lower bounds for the bearing capacity are taken, which are available in the reference in the following form:

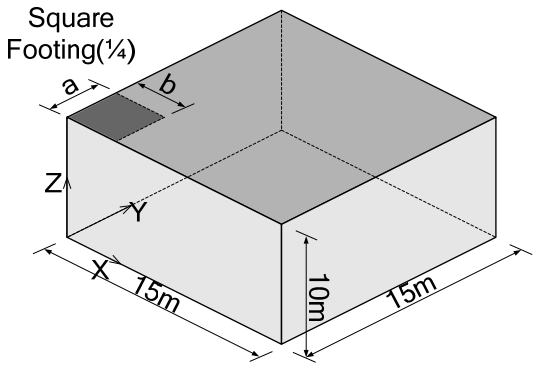
$$q_{\max} = c\left(5.24 + 0.47\frac{a}{b}\right) \quad \frac{a}{b} \geq 0.53$$

$$q_{\max} = c\left(5.14 + 0.66\frac{a}{b}\right) \quad \frac{a}{b} < 0.53$$

$$q_{\min} = c(2+\pi)$$

where c is the cohesion of the material.

Figure 3.10.1
Quarter symmetry model of square footing problem



Material data	Young's modulus	*E = 257142.9 kPa*
	Poisson's ratio	*ν = 0.285714*
	Yield criteria	*Mohr-Coulomb*
	Cohesion	*c=100 kPa*
	Friction angle	$\phi = 0°$
	Dilatancy angle	$\psi = 0°$

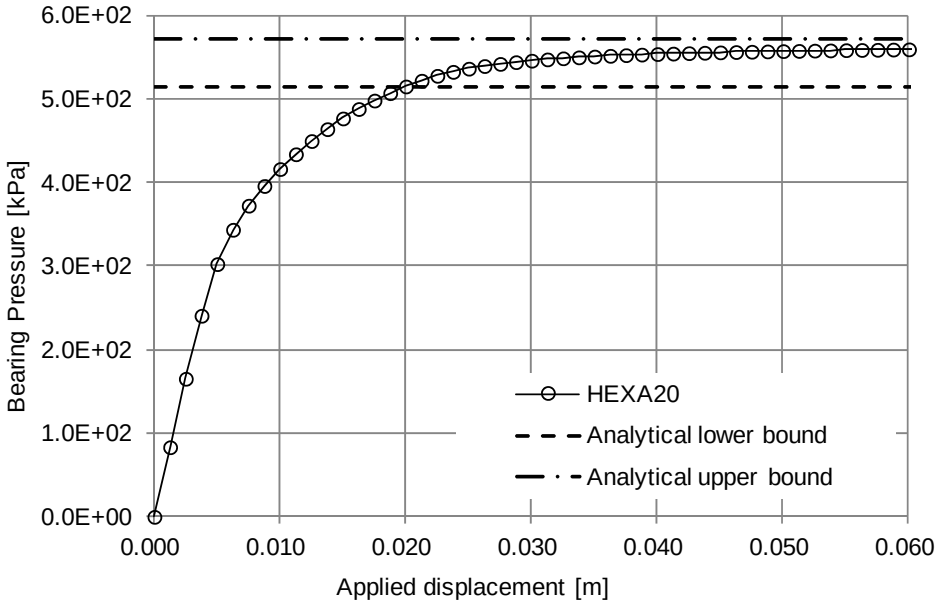


Figure 3.10.2
Bearing pressure vs. applied displacement for square footing problem obtained using solid elements

Table 3.10.1 Bearing capacity of square footing problem (F_Z)

Collapse Load	F_Z [kPA] at upper bound	F_Z [kPA] at lower bound
Reference	*571.0*	*514.2*
HEXA-20	*560.3*	

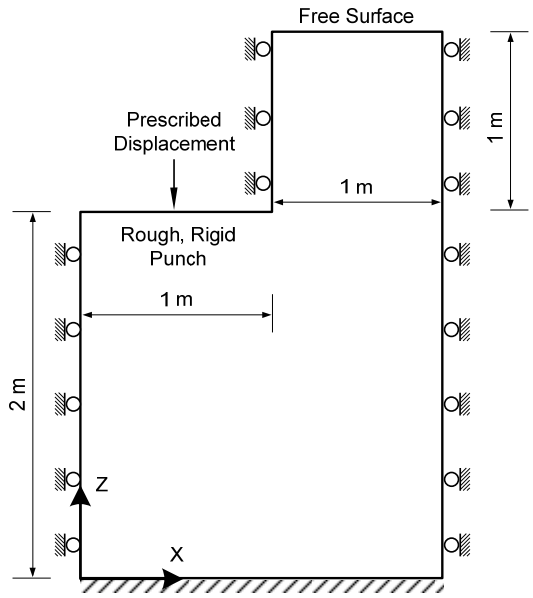
3.11 Rigid Punch Penetrating into Clay

REFERENCE	Van Langen, H. et al. [12]
ELEMENTS	Plane strain elements, Footing, Cohesion varing
MODEL FILENAME	Material11.gts

Figure 3.11.1 depicts a rigid punch penetrating into clay. The base of the punch is assumed to be fully rough whereas the side is fully smooth. The constitutive behavior of the material is elastic-perfectly plastic Tresca model. Nonlinear static analysis is carried out with the punch penetration modeled as vertical displacements prescribed at the base of the punch. The rough base is taken into account by constraining horizontal displacements at the base of the punch.

The resulting limit pressure is compared with an analytical solution based on slip line field theory. The limiting pressure for the dimension shown in Figure 3.11.1 is expressed as, $p = (\pi + 2)c$ where c is the cohesion of the material.

Figure 3.11.1
Geometry and boundary conditions of the rigid punch penetration problem



Material data	Young's modulus	*E = 2500 kPa*
	Poisson's ratio	*ν = 0.25*
	Yield criteria	*Mohr-Coulomb*
	Cohesion	*c=10 kPa*
	Friction angle	$\phi = 0°$
	Dilatancy angle	$\psi = 0°$

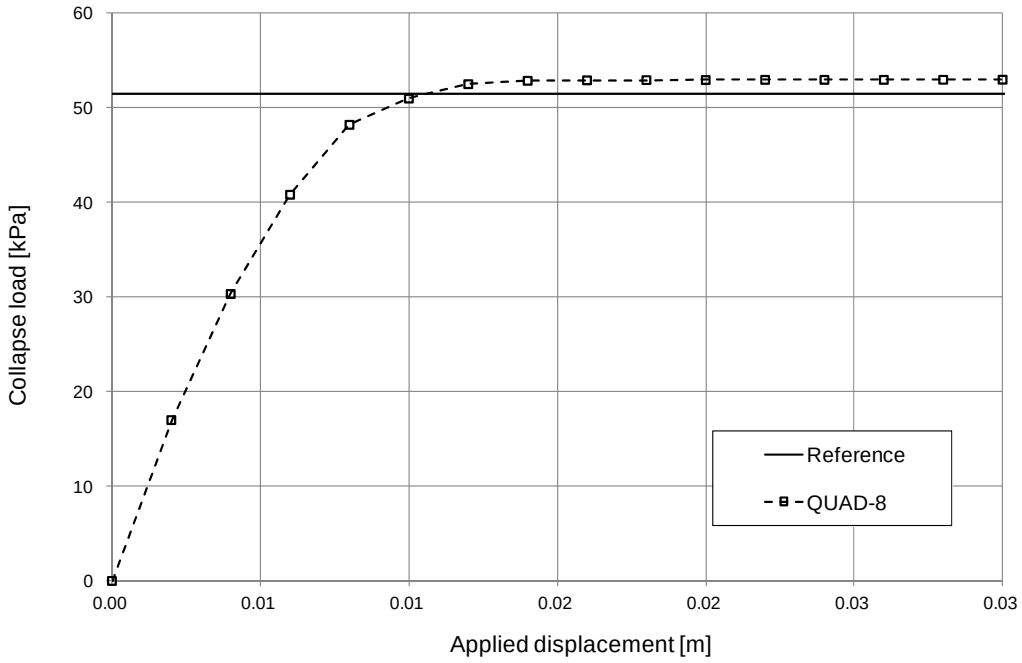


Figure 3.11.2
Average pressure vs. applied displacement obtained using plane strain elements

Table 3.11.1 Collapse load for 8-element punch

Collapse load	F_Z [kN] at punch
Reference	*51.42*
QUAD-8	*52.97*

3.12 Cylindrical Hole in an Infinite Mohr-Coulomb Medium

REFERENCE	Salencon, J [13]
ELEMENTS	Plane strain elements, axisymmetric elements
MODEL FILENAME	Material12_1.gts, Material12_2.gts

Cylindrical hole in an infinite medium is considered. The infinite medium is subjected to in-situ normal stress of -30MPa aligned with the axes as shown in Figure 3.12.1. The constitutive behavior of the material is elastic-perfectly plastic Mohr-Coulomb model. Nonlinear static analysis is carried out with both plane strain and axisymmetric elements. The plane strain approach models the infinite medium as a square with side length of 20m and discretizes quarter of the model utilizing symmetry. The axisymmetric approach models the medium as a cylinder with radius of 10 m. The distribution of stresses and displacements along the radius are compared with an analytical solution.

Figure 3.12.1
Quarter model of cylindrical hole

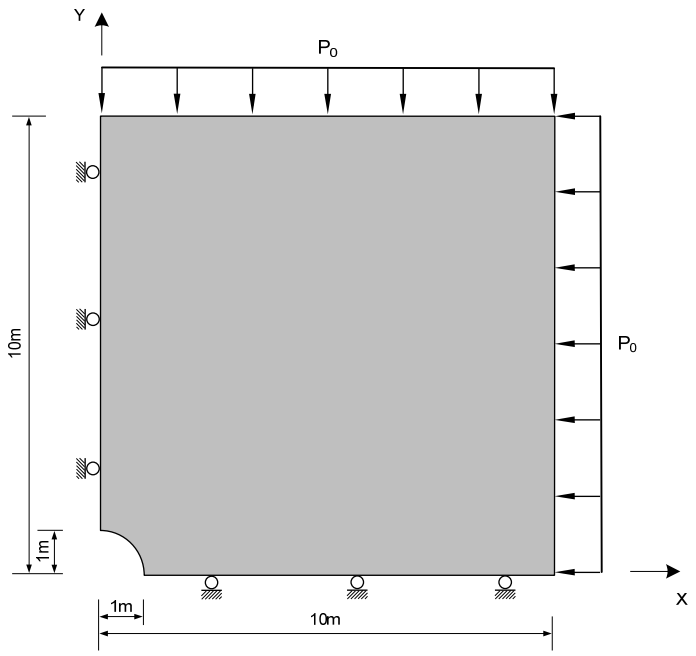
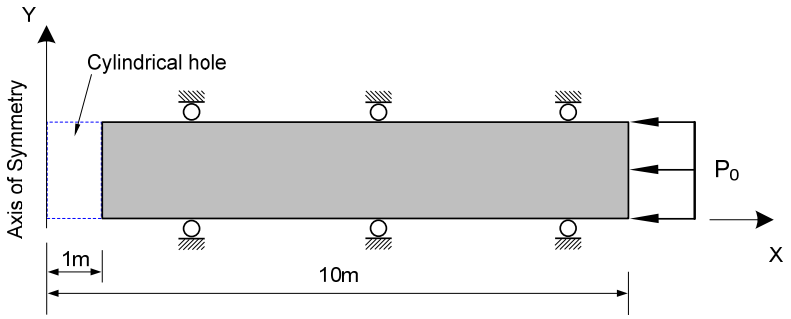


Figure 3.12.2
Axisymmetric model of cylindrical hole



Material data	Young's modulus	*E = 6777.931 MPa*
	Poisson's ratio	*ν = 0.210345*
	Yield criteria	*Mohr-Coulomb*
	Cohesion	*c=3.45 MPa*
	Friction angle	$\phi = 30°$
	Dilatancy angle	$\psi = 30°$
Load data	Prestress	*-30 Mpa*
	Edge pressure	*30 Mpa*

Figure 3.12.3
Normalized stress vs. radius obtained using plane strain elements

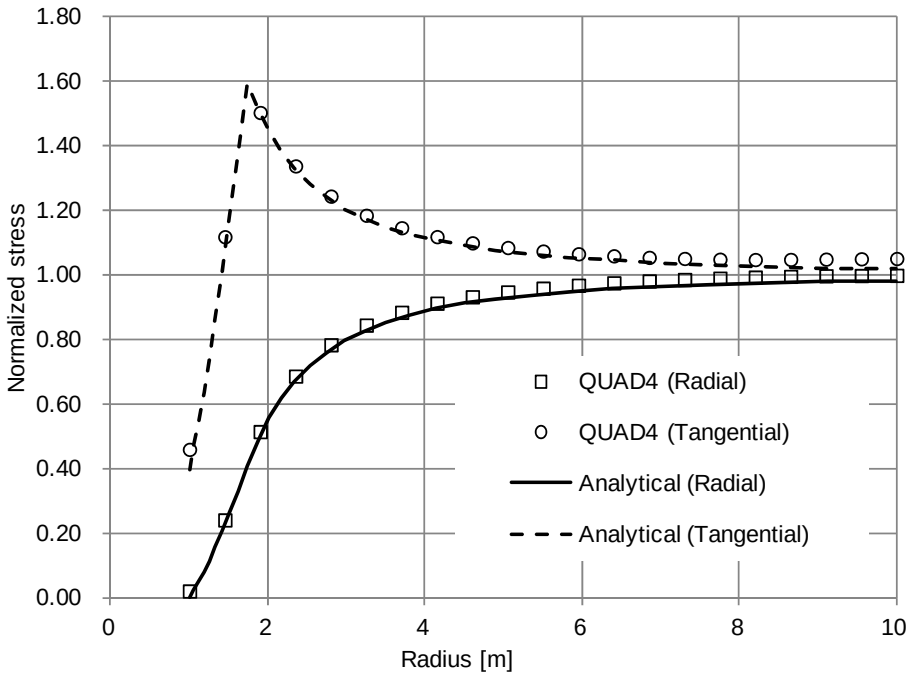


Figure 3.12.4
Normalized displacement vs. radius obtained using plane strain elements

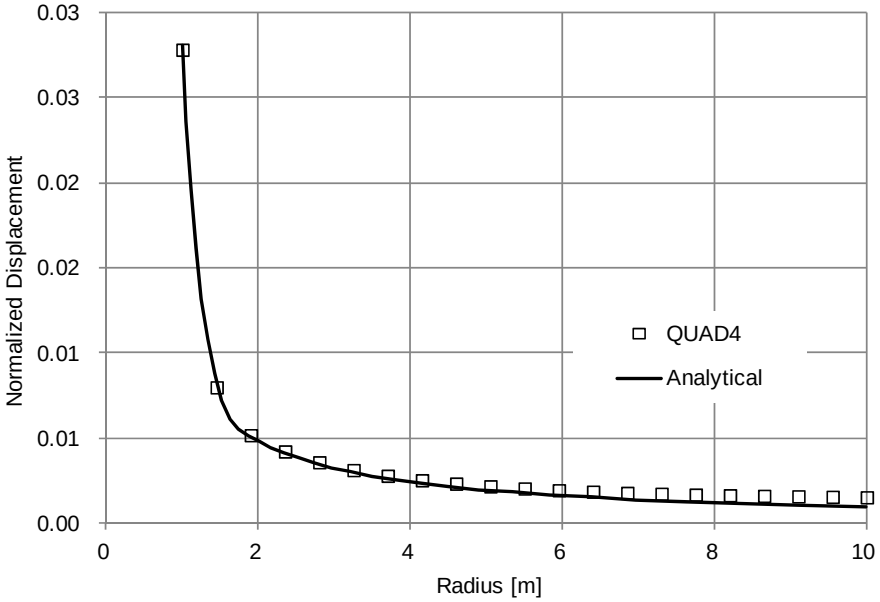
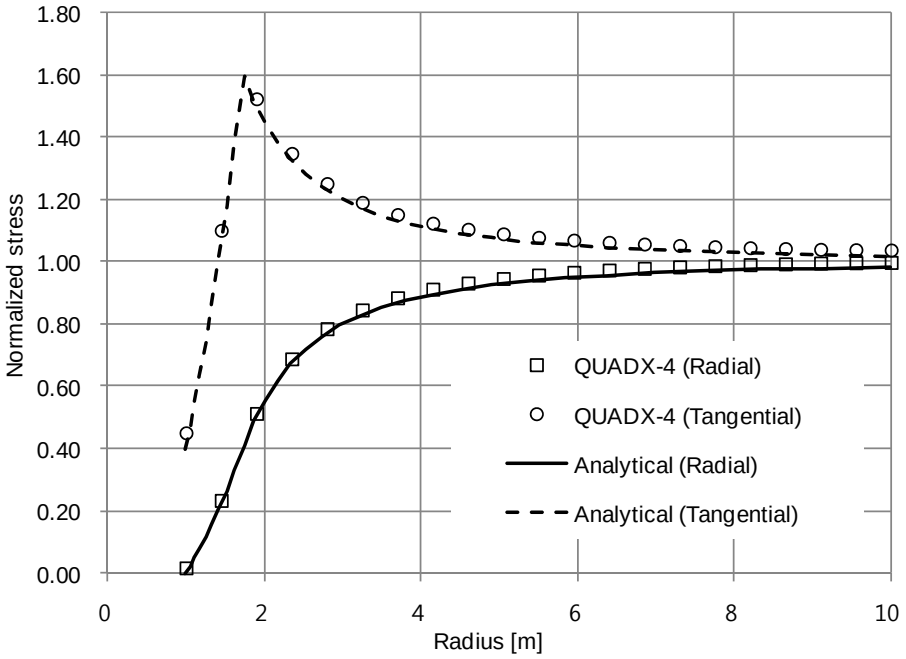


Figure 3.12.5
Normalized stress vs. radius obtained using axisymmetric elements



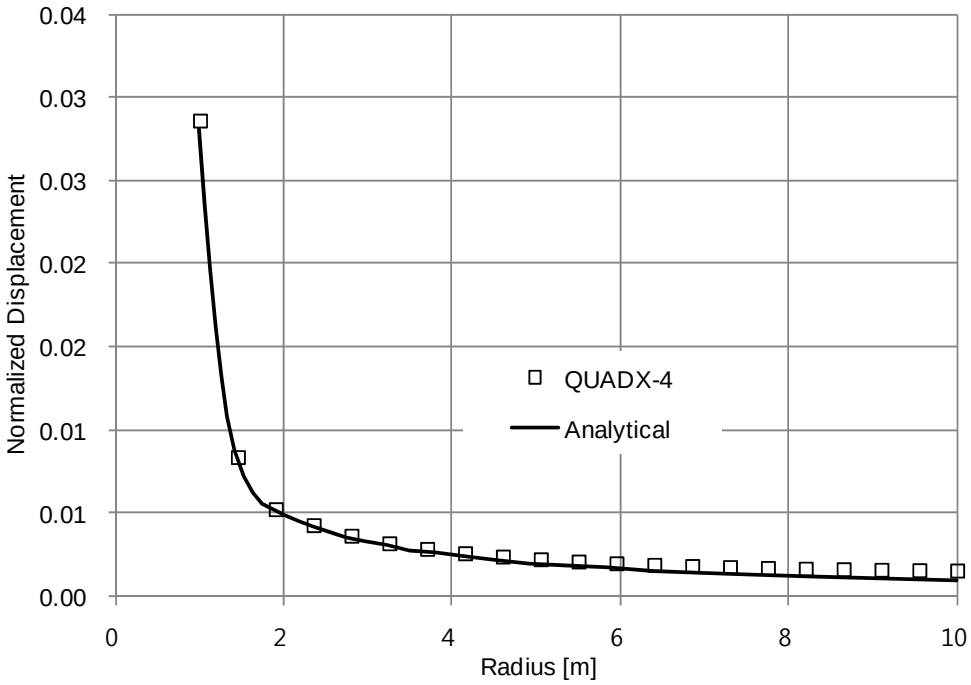


Figure 3.12.6
Normalized displacement vs. radius obtained using axisymmetric elements

Table 3.12.1 Yield zone radius

	Yield zone radius [m]
Reference	*1.731*
QUAD-4	*1.81*
QUADX-4	*1.81*

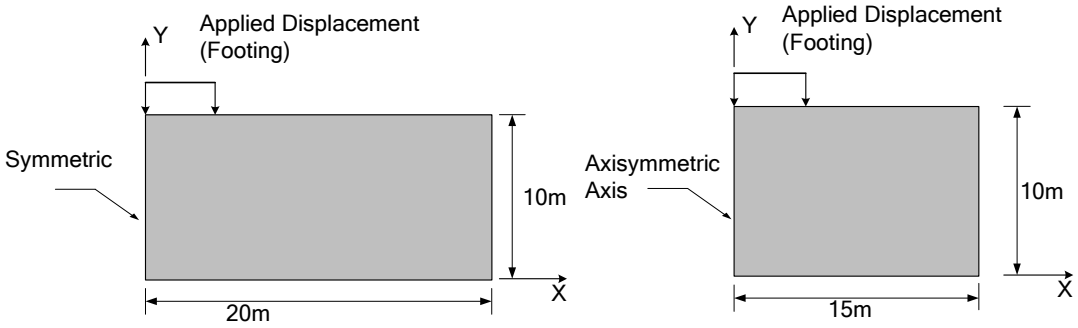
3.13 Strip and Circular Footings on a Mohr-Coulomb Soil

REFERENCE	Terzaghi, K. et al.[14], Cox et al. [15]
ELEMENTS	Plane strain elements, solid elements
MODEL FILENAME	Material13_1.gts, Material13_2.gts, Material13_3.gts, Material13_4.gts

Bearing capacity of Mohr-Coulomb soil is evaluated in this problem. Two types of footing conditions are considered: a strip footing and a circular footing. Plane strain and solid elements are utilized to solve the footing problem. Axisymmetric and solid elements are employed for the circular footing problem. Prescribed displacements in the footing area are gradually increased to obtain the limiting footing pressure. Analytic solutions are taken for the standard of comparison.

The bearing pressure for the strip footing problem with friction angle of soil equal to zero can be expressed as $q=(2+\pi)c$, where c is the cohesion of the material. For the circular footing problem, the semi-analytical solution of the average pressure over the footing at failure for a friction angle of 20° is expressed as $q = 20.1c$.

Figure 3.13.1 Schematic model for strip and circular footing problem



(a) Strip footing

(b) Circular footing

Material data (strip footing)	Young's modulus	*E = 257140 kPa*
	Poisson's ratio	*ν = 0.285714*
	Yield criteria	*Mohr-Coulomb*
	Cohesion	*c=100 kPa*
	Friction angle	$\phi = 0°$
	Dilatancy angle	$\psi = 0°$

Material data (circular footing)	Young's modulus	*E = 257140 kPa*
	Poisson's ratio	*ν = 0.285714*
	Yield criteria	*Mohr-Coulomb*
	Cohesion	*c=100 kPa*
	Friction angle	$\phi = 20°$
	Dilatancy angle	$\psi = 20°$

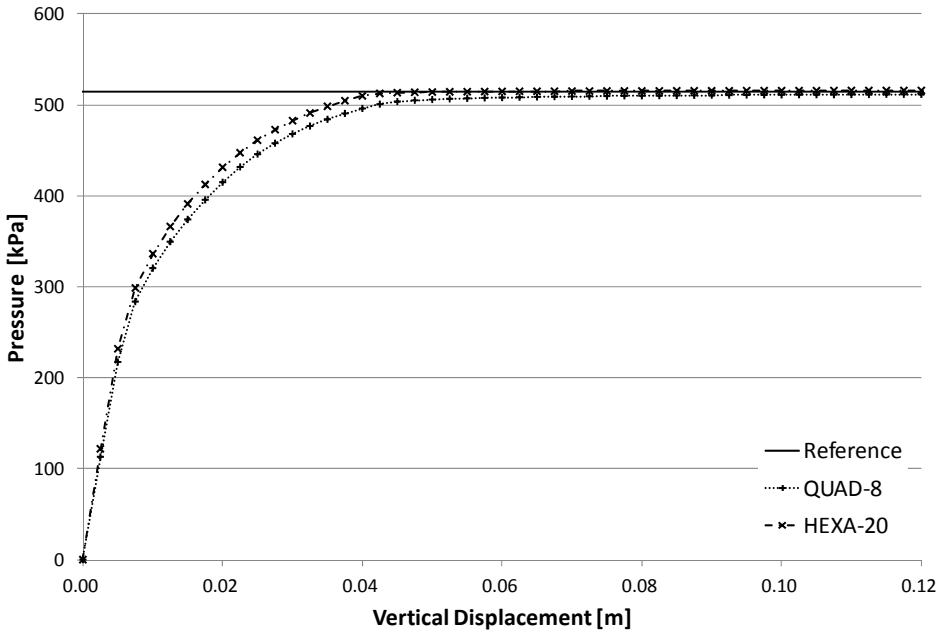


Figure 3.13.2
Pressure vs. vertical displacement response of strip footing problem obtained using plane strain and solid elements

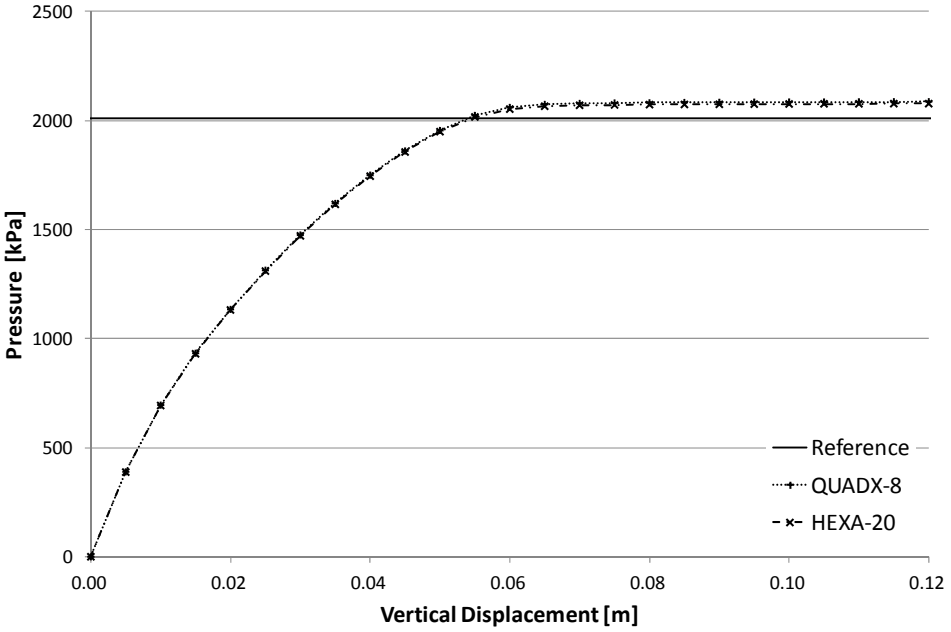


Figure 3.13.3
Pressure vs. vertical displacement response of circular footing problem obtained using axisymmetric and solid elements

Table 3.13.1 Bearing Capacity of Strip and Circular footings in KPa.

	Strip Footing	Circular Footing
Reference	*514.2*	*2010.0*
QUAD-8	*511.62*	-
QUADX-8	-	*2090.3*
HEXA20	*515.81*	*2081.1*

3.14 Effect of Strain Rate on Undrained Effective Stress Path and Stress-Strain Response

REFERENCE	Sekiguchi, H., et al.[16]
ELEMENTS	Solid elements
MODEL FILENAME	Material14_1.gts, Material14_2.gts

Figure 3.14.1 depicts undrained compression and extension simulations carried out under K0 consolidated triaxial conditions. Vertical pressure is applied on the upper surface and K0 condition is applied to reproduce in-situ stresses at initial stage. In the subsequent stage, specified displacement loads are applied in the z-direction with various strain rates to demonstrate the time effects on undrained responses.

Effective stress paths and stress-strain responses are plotted and compared with those from the original Sekiguchi-Ohta model in Figures 3.14.2 and 3.14.3. Dots represent the reference data and the lines are those obtained from the Sekiguchi-Ohta inviscid and viscid models in GTS NX.

Shown in Figure 3.14.2, the undrained strengths, defined as the maximum value of $|\sigma_a - \sigma_r|/2$, are dependent on the strain rates. Also, the paths are asymmetric for compression and extension sides, as are also evident from the reference graphs.

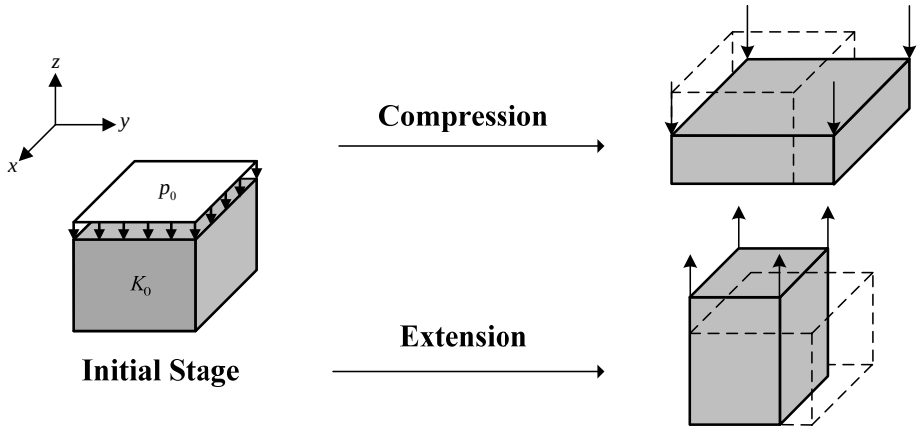


Figure 3.14.1 triaxial compression and extension test configuration

Material data	Poisson ratio	ν = 0.3
	Slope of consolidation line	$\lambda/(1+e_0)$ =0.0921
	Slope of over-consolidation line	$\kappa/(1+e_0)$ =0.0199
	Coefficient of dilatancy	D = 0.053
	Secondary compression Coefficient	α = 0.0029
	Shear modulus	G/p_0 = 25
	Initial volumetric strain ratio	$\dot{v}_0$ =10^{-7}
	K_0	K_0 = 0.5

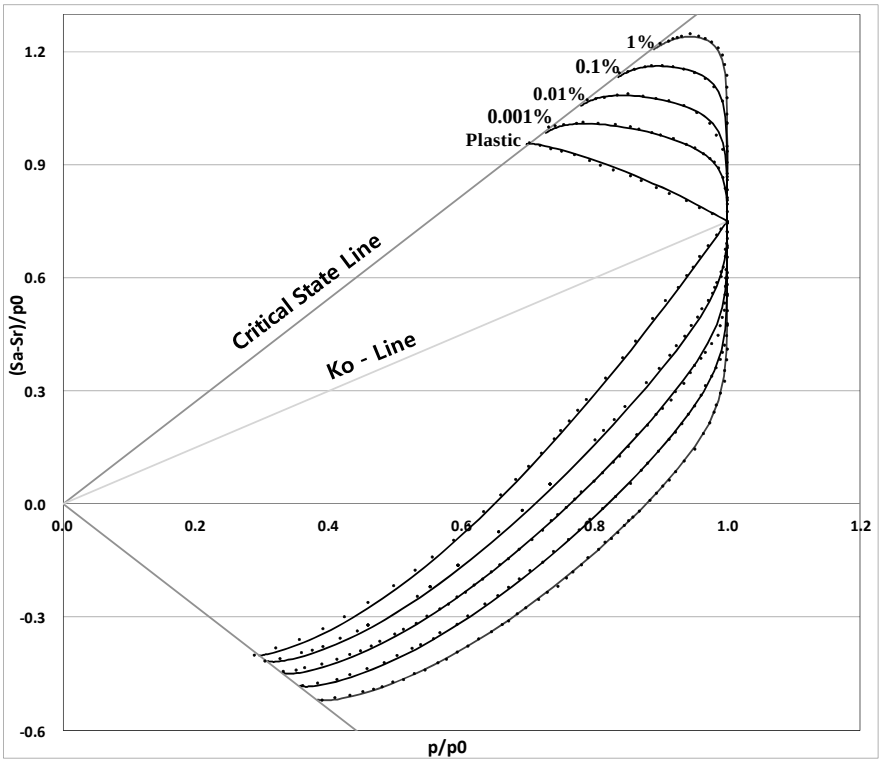


Figure 3.14.2
Undrained effective stress paths with different strain rates

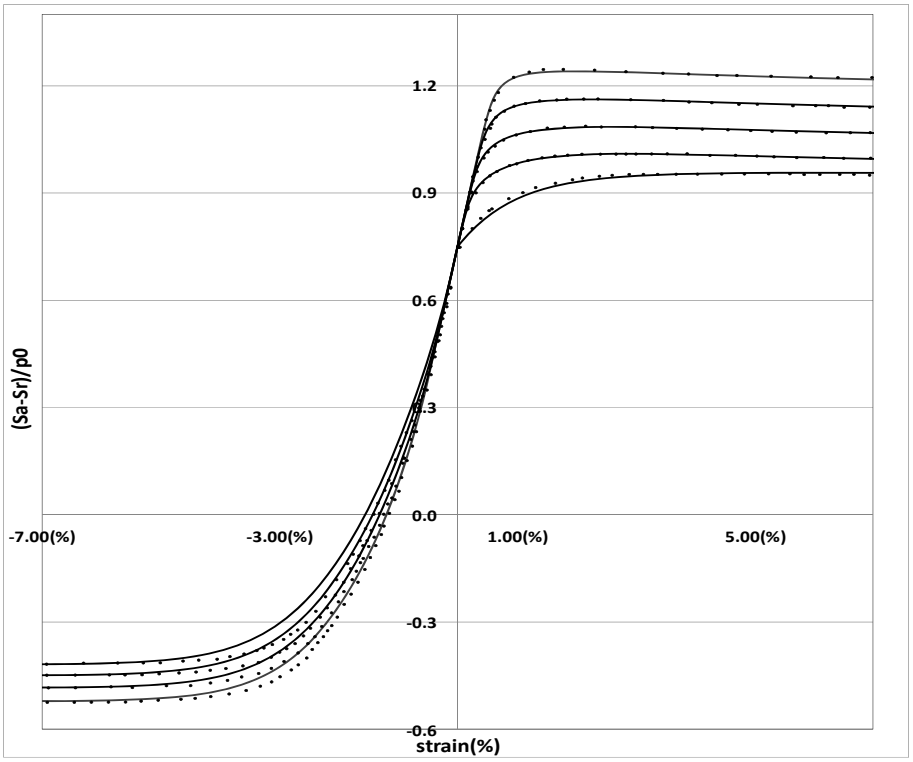


Figure 3.14.3
Undrained stress-strain response with different strain rates

3.15 Oedometer Test for Hardening Soil with Small Strain Stiffness with Drained Condition

REFERENCE	Benz, T.[17]
ELEMENTS	Solid elements
MODEL FILENAME	Material15_1.gts, Material15_2.gts

Oedometer compression tests are carried out using "Hardening soil" material model and "Hardening soil with small strain stiffness" material model with drained condition. Figure 3.15.1 depicts the test configuration and the specimen has unit dimension.

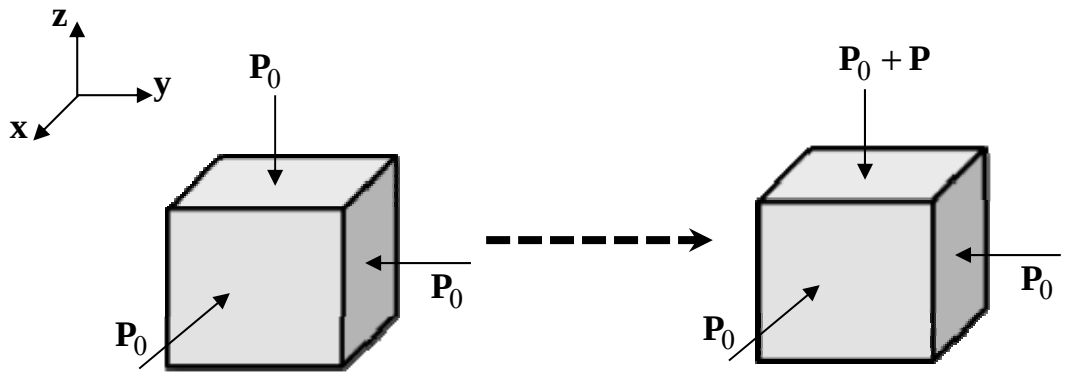
The material table below shows the data of loose and dense "Hostun" sand, used in the simulation. The only differences of "Hardening soil" and "Hardening soil with small strain stiffness" model are "shear modulus at small strain (G0ref)" and "threshold shear strain".

The simulations are carried out with two different material properties. At initial stage, the specimens are loaded with uniform confining pressure 25kPa. In the case of loose "Hostun" sand, the additional pressure along z direction is loaded consecutively to 50kPa, 100kPa, 200kPa with intermediate unloading stages. In the case of dense "Hostun" sand, the additional pressure along z direction is loaded consecutively to 100kPa, 200kPa, 400kPa with intermediate unloading stages.

Figure 3.15.2, 3.15.3 show the results of simulation. The result of the loose "Hostun" sand are shown in the figure 3.16.2 and one of the dense "Hostun" sand are shown in the figure 3.16.2.

The test results show that the "Hardening soil with small strain stiffness" model produce a little stiffer response than "Hardening soil" model. And due to high "threshold shear strain", the difference between models can be seen more clearly in dense sand.

Figure 3.15.1
Oedometer test configuration



Material data	Hostun sand	Loose	*Dense*
	Poisson ratio	0.2	0.2
	E50 ref	20000 kPa	40000 kPa
	Eoed ref	20000 kPa	40000 kPa
	Eur ref	60000 kPa	120000 kPa
	Failure ratio	0.9	0.9
	Reference pressure	100 kPa	100 kPa
	Power of stress level	0.5	0.5
	K0nc	0.5	0.36
	Friction angle	30. degree	40. degree
	Dilatancy angle	0. degree	10. degree
	Cohesion	0.	0.
	Tensile strength	0. kPa	0. kPa
	G0 ref	70000 kPa	122500 kPa
	Threshold shear strain	0.0001	0.0002

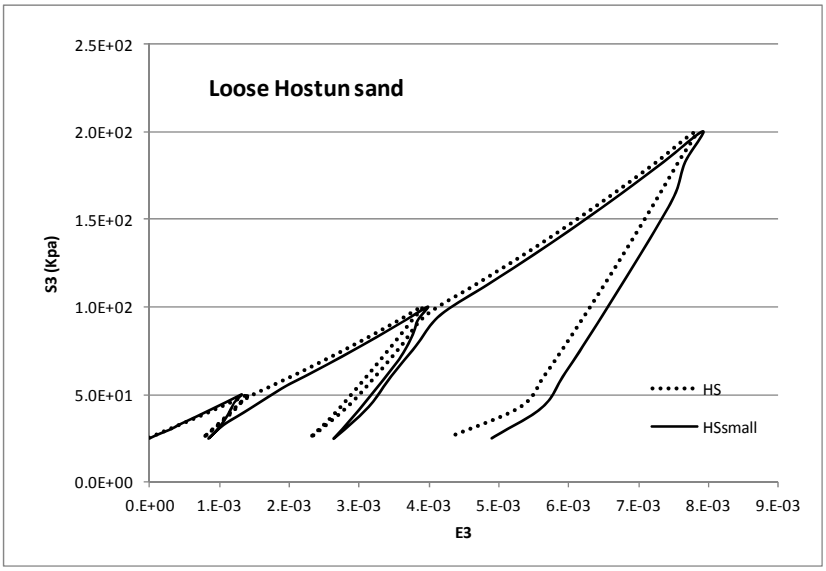


Figure 3.15.2
Oedometer test on loose Hostun sand

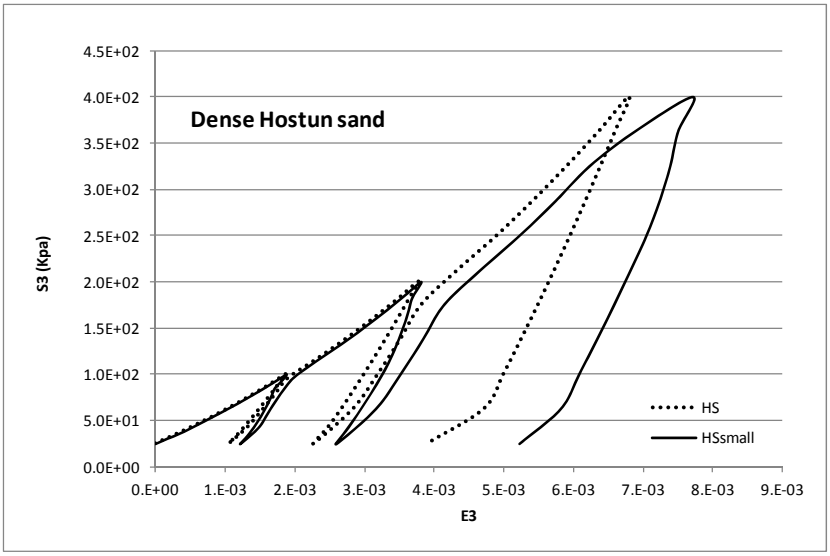


Figure 3.15.3
Oedometer test on dense Hostun sand

3.16 Triaxial Test for Hardening Soil with Small Strain Stiffness with Drained Condition

REFERENCE	Benz, T.[17]
ELEMENTS	Solid elements
MODEL FILENAME	Material16_1.gts, Material16_2.gts

Triaxial compression tests are carried out using "Hardening soil" material model and "Hardening soil with small strain stiffness" material model with drained condition. Figure 3.16.1 depicts the test configuration and the specimen has unit dimension.

The material table below shows the data of loose and dense "Hostun" sand, used in the simulation. The only differences of "Hardening soil" and "Hardening soil with small strain stiffness" model are "shear modulus at small strain (G0ref)" and "threshold shear strain".

The simulations are carried out with two different material properties and three different initial conditions.. At initial stage, the specimens are loaded with uniform confining pressure 100kPa, 300kPa, 600kPa respectively. And in the subsequent stage, prescribed displacements are loaded along the z-direction.

According to the specification of the material model, the hyperbolic relation between the vertical strain and the deviatroic stress, "q", in triaxial test only works if the "q" is less than the ultimate deviatoric stress, "qf", which comes from the mohr-coulomb model. And if the "q" is more than the "qf", the models works like perfect plastic mohr-columb model.

Figure 3.16.2, 3.16.3, 3.16.4, 3.16.5 show the results of simulation. The results of the loose "Hostun" sand are shown in the figure 3.16.2, 3.16.3, and ones of the dense "Hostun" sand are shown in the figure 3.16.4, 3.16.5.

The axial strain vs. stress ratio is compared with the analytical solution of "Hardening soil" model in the figure 3.16.2, 3.16.4. and the analytical solutions are only calculated up to the ultimate deviatoric stress. In the figures, "stress ratio" means the ratio of loading stress along z-direction to confining stress. From the results, it is obvious that perfect yielding occurs above the ultimate deviatoric stress.

The "dense sand" has dilatancy angle and the "loose sand" doesn't. The dilatancy effect can be verified from the results of axial strain vs. volumetric strain shown in the figure 3.16.3, 3.16.5.

All test results show that there are little difference between "Hardening soil" model and "Hardening soil with small strain stiffness" model in triaxial test.

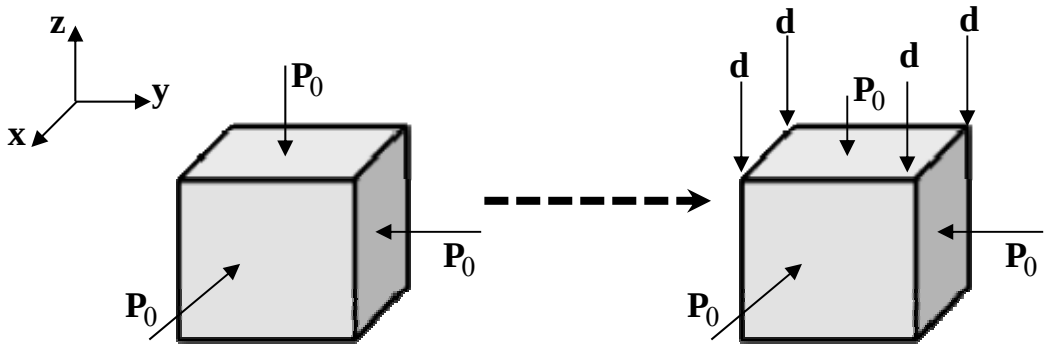


Figure 3.16.1 Triaxial test configuration

Material data	Hostun sand	Loose	*Dense*
	Poisson ratio	0.2	0.2
	E50 ref	20000 kPa	40000 kPa
	Eoed ref	20000 kPa	40000 kPa
	Eur ref	60000 kPa	120000 kPa
	Failure ratio	0.9	0.9
	Reference pressure	100 kPa	100 kPa
	Power of stress level	0.5	0.5
	K0nc	0.5	0.36
	Friction angle	30. degree	40. degree
	Dilatancy angle	0. degree	10. degree
	Cohesion	0.	0.
	Tensile strength	0. kPa	0. kPa
	G0 ref	70000 kPa	122500 kPa
	Threshold shear strain	0.0001	0.0002

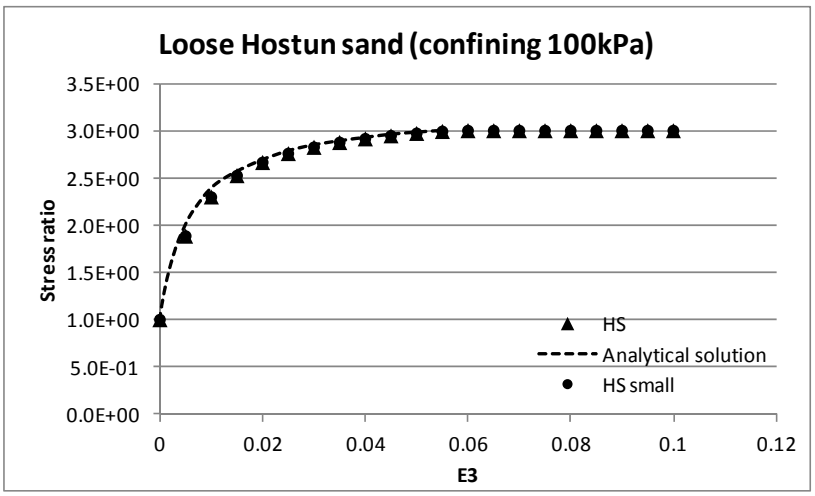


Figure 3.16.2.(a) Axial strain vs. stress ratio of the loose Hostun sand with confining stress 100kPa

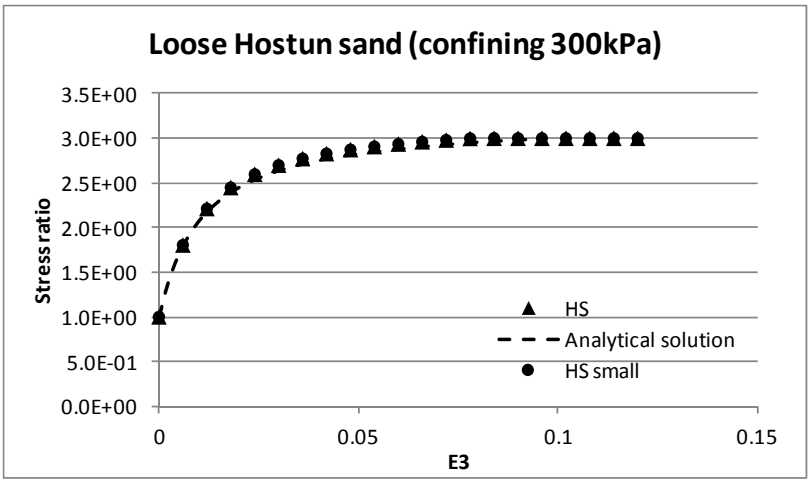


Figure 3.16.2.(b) Axial strain vs. stress ratio of the loose Hostun sand with confining stress 300kPa

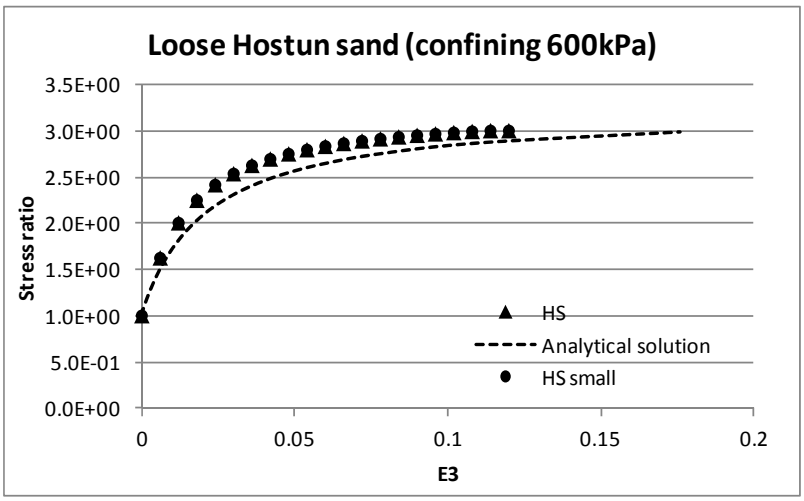


Figure 3.16.2.(c) Axial strain vs. stress ratio of the loose Hostun sand with confining stress 600kPa

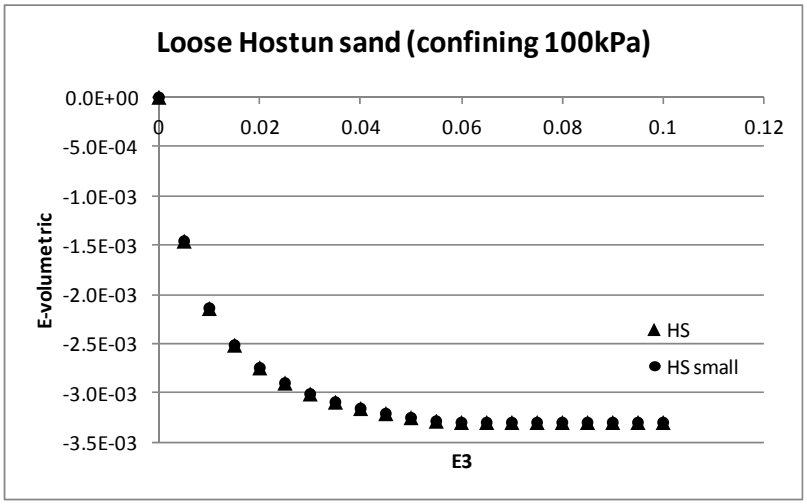


Figure 3.16.3.(a)
Axial strain vs. volumetric strain of the loose Hostun sand with confining stress 100kPa

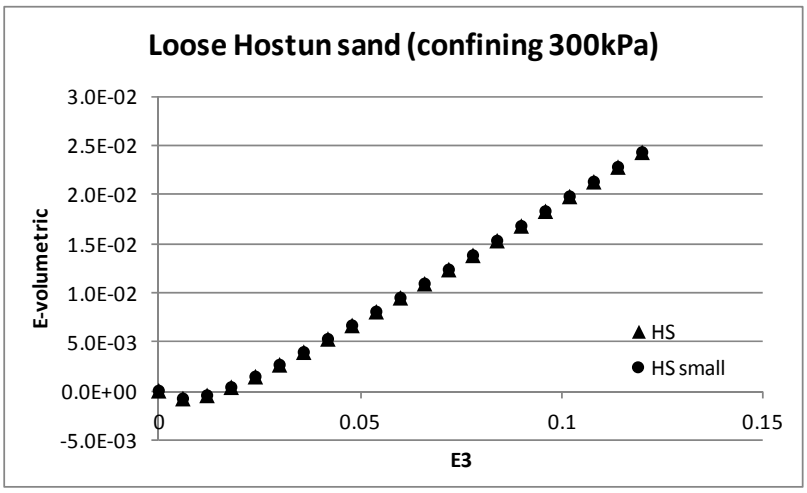


Figure 3.16.3.(b)
Axial strain vs. volumetric strain of the loose Hostun sand with confining stress 300kPa

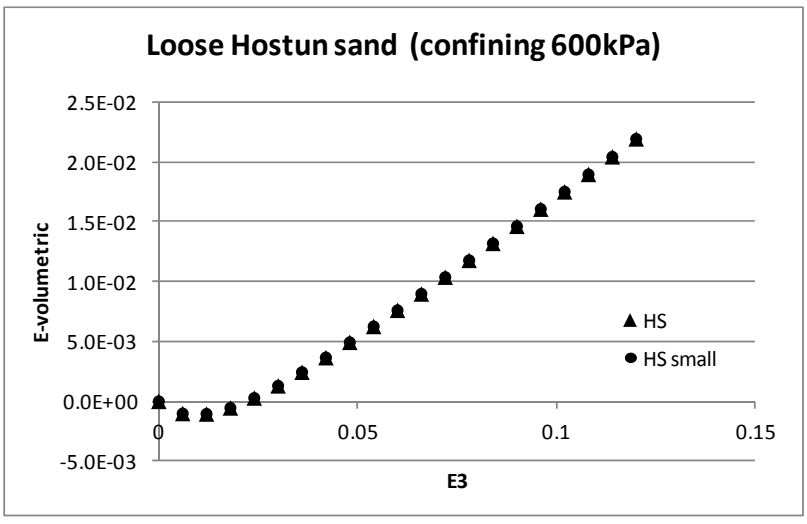


Figure 3.16.3.(c)
Axial strain vs. volumetric strain of the loose Hostun sand with confining stress 600kPa

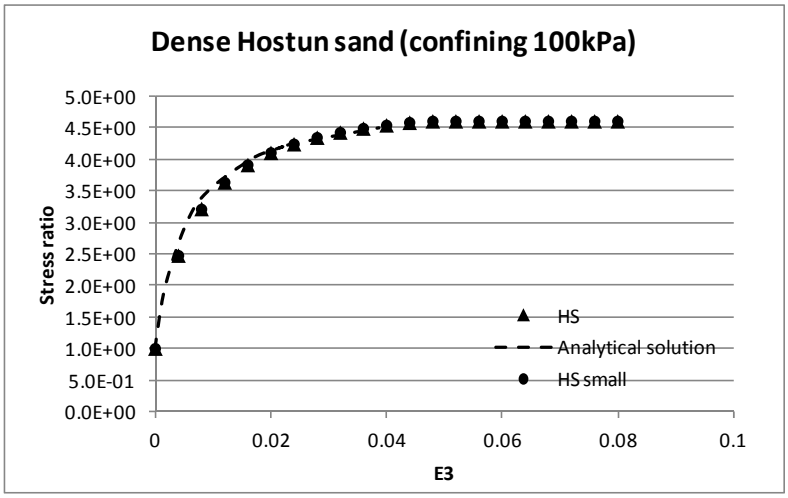


Figure 3.16.4.(a)
Axial strain vs. stress ratio of the dense Hostun sand with confining stress 100kPa

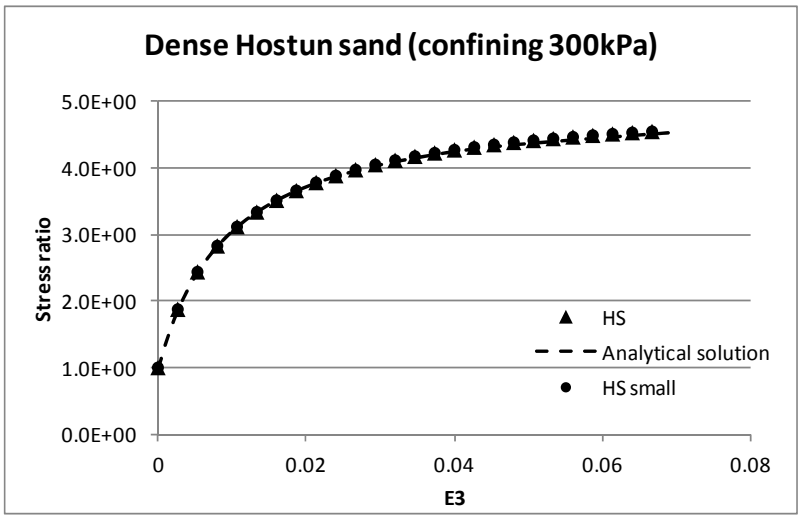


Figure 3.16.4.(b)
Axial strain vs. stress ratio of the dense Hostun sand with confining stress 300kPa

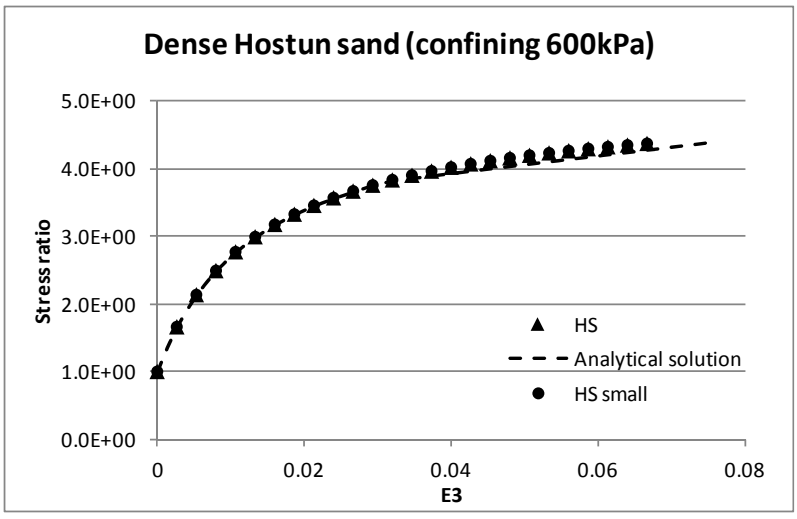


Figure 3.16.4.(c)
Axial strain vs. stress ratio of the dense Hostun sand with confining stress 600kPa

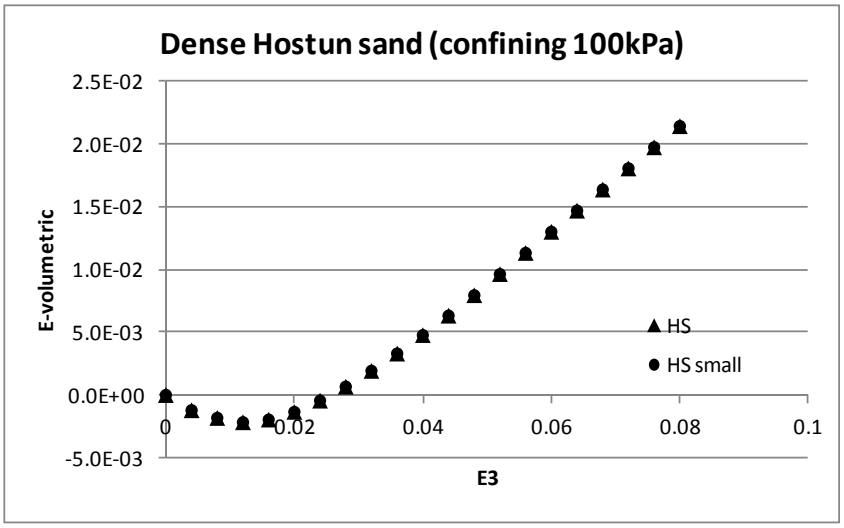


Figure 3.16.5.(a)
Axial strain vs. volumetric strain of the dense Hostun sand with confining stress 100kPa

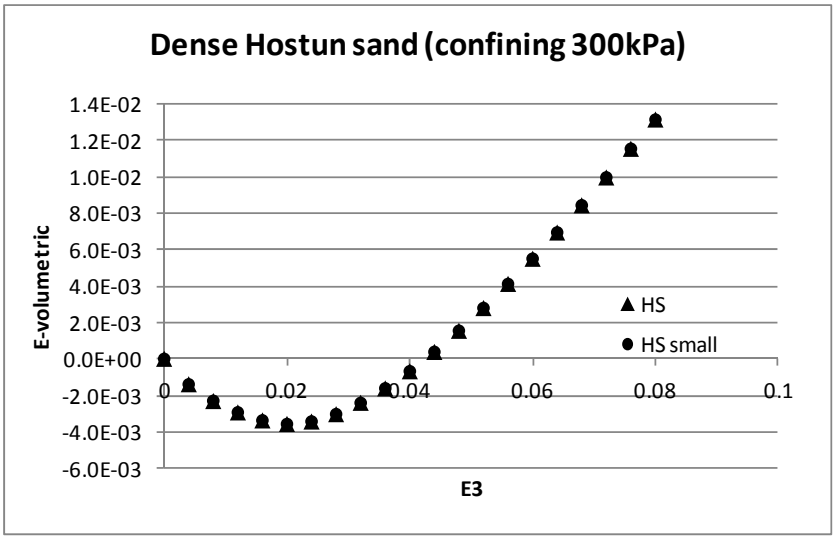


Figure 3.16.5.(b)
Axial strain vs. volumetric strain of the dense Hostun sand with confining stress 300kPa

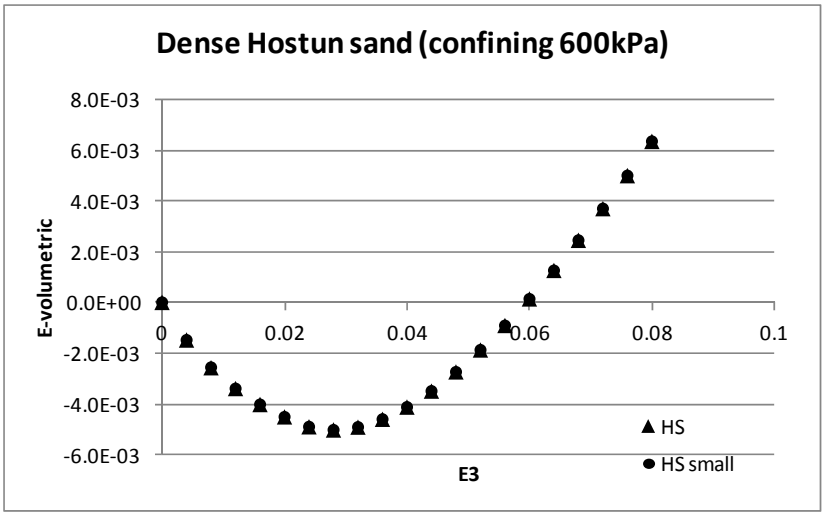


Figure 3.16.3.(c)
Axial strain vs. volumetric strain of the dense Hostun sand with confining stress 600kPa

3.17 Undrained Triaxial Test with SCLAY1 Material Model

REFERENCE	LEONI, M.[18]
ELEMENTS	Solid element
MODEL FILENAME	Material17_1.gts

SCLAY1 is an elasto-plastic model which accounts for initial stress induced anisotropy and rotational hardening law. Generalized SCLAY1S model available in GTS NX is an advanced version of SCLAY1 model that also accounts for bonding and destructuration law. Generalized SCLAY1S model can be degenerated into SCLAY1 model when parameters related bonding and destructuration law are set to zero.

Undrained triaxial compression and extension simulations are carried out with degenerated Generalized SCLAY1S model to focus on the effect of rotational hardening law. Initially, vertical pressure is applied on the sample and in-situ stresses are generated with K0 condition. At the subsequent stage, specified displacements in the z-direction are applied to model the compression and extension tests.

Effective stress paths and stress-strain behavior of compression and extension tests are obtained. The results are compared with those from the reference. In Figures 3.17.2 and 3.17.3, the lines represent the data obtained using GTS NX and dots are the reference data. Characteristic responses of the SCLAY1S model are well captured by using GTS NX. Asymmetry of stress paths as depicted in Figure 3.17.2 are mainly attributed to initial stress induced anisotropy. The 'hook' type stress path is usually recognized as a defect of the model in undrained triaxial extension.

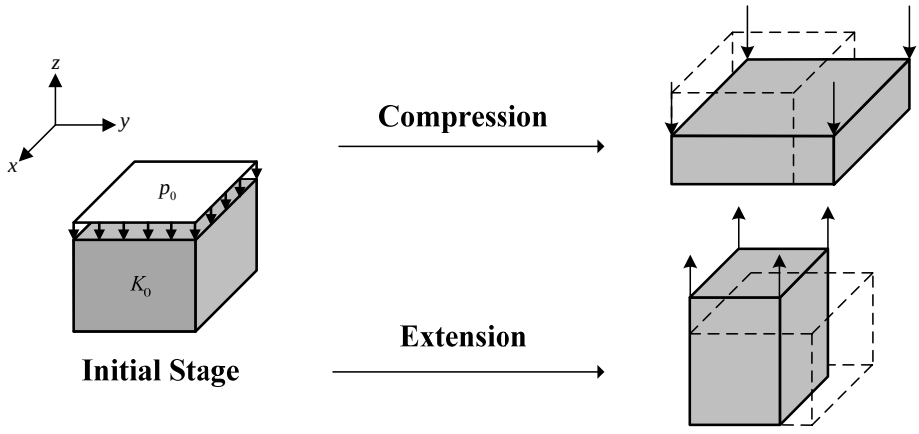


Figure 3.17.1. triaxial compression / extension test with K0 condition

Material data	Poisson ratio	$\nu = 0.255$
	Slope of consolidation line	$\lambda/(1+e_0) = 0.1055$
	Slope of over-consolidation line	$\kappa/(1+e_0) = 0.0161$
	Slope of critical state line	$M = 1.29$
	Degree of anisotropy	$\alpha = 0.493$
	Abs. effectiveness of rotational hardening	$\mu = 28$
	Rel. effectiveness of rotational hardening	$\beta = 0.856$
	K0	$K_0 = 0.47$

Figure 3.17.2
Undrained effective stress paths

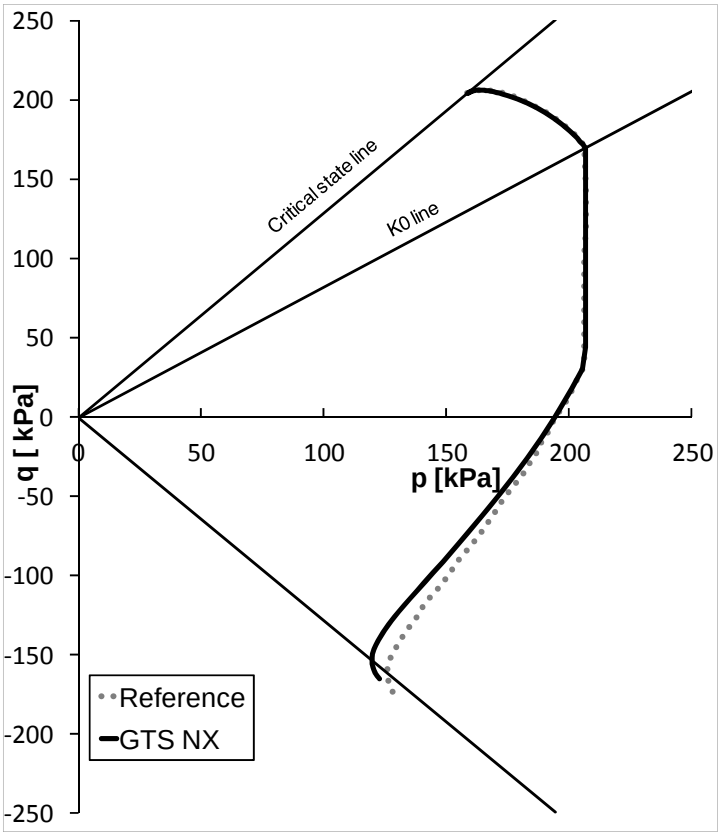
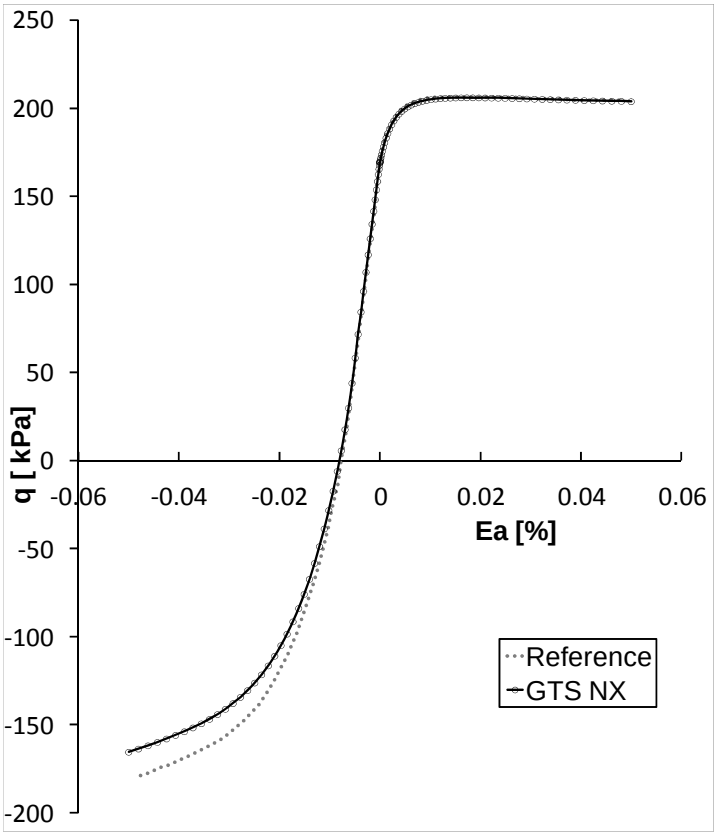


Figure 3.17.3
Undrained stress-strain responses



3.18 Unit Tests of Coulomb Friction Model for Interface Elements

REFERENCE	Analytical solution
ELEMENTS	Line interface elements, Plane strain elements
MODEL FILENAME	Material18_1.gts, Material18_2.gts

Unit tests are carried out for the coulomb friction model of interface elements.

Figure 3.18.1 depicts the test configuration. In the figure, two quadrilateral blocks mean soil and between them, there are interface elements to correlate. The gap between soil has no physical meaning, but exist for the description of clarity of interface elements.

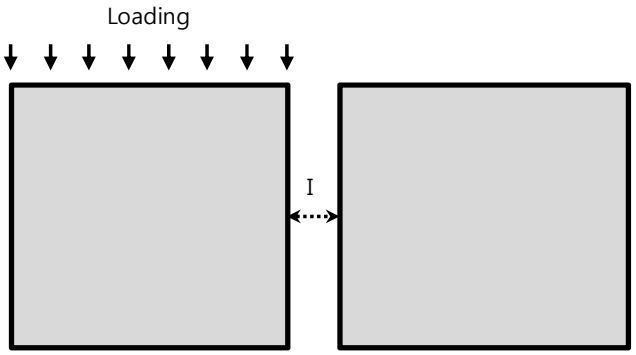


Figure 3.18.1.
Unit test configuration

The column friction model is defined as the following formula and for more details, review the reference manual.

$$f = \sqrt{\mathbf{t}_t^2} + t_n \tan\phi(\kappa) - c(\kappa) = 0$$

$$g = \sqrt{\mathbf{t}_t^2} + t_n \tan\psi$$

The table below show the material data for interface elements.

Material data for pile	Normal stiffness modulus	K_n = 50. kN/m^3
	Shear stiffness modulus	K_t = 40. kN/m^3
	Friction angle	ϕ = 30. degree
	Dilatancy angle	ψ = 0. / 10. degree
	Cohesion	c = 0.04 kN/m^3
Material data for pile tip	Normal stiffness modulus	K_n = 50. kN/m^3
	Shear stiffness modulus	K_t = 40. kN/m^3
	Friction angle	ϕ = 30. degree
	Dilatancy angle	ψ = 0. / 10. degree
	Cohesion	

Plastic displacement	Cohesion (kN/m^3)
0.0	0.04
0.2	1.0

Interface property	Thickness	1.0 m

The simulations are carried out with respect to the constant cohesion and the plastic dependent cohesion with two dilatancy angles respectively. Soil is loaded using self weight at initial stage and to induce shear stress, the left side soil is loaded with a specified displacement at the subsequent stage.

For the following figure, probing points for interface element results are selected at the 1st node of interface element at the top side soil.

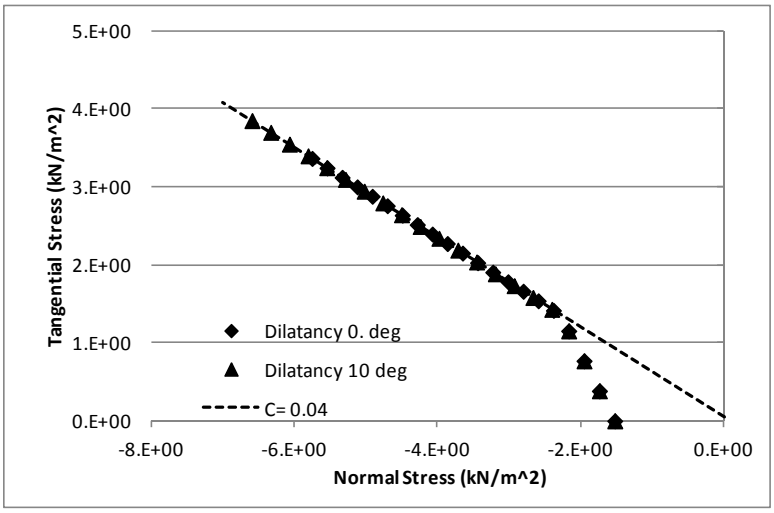


Figure 3.18.2
Unit test for interface elements with constant cohesion

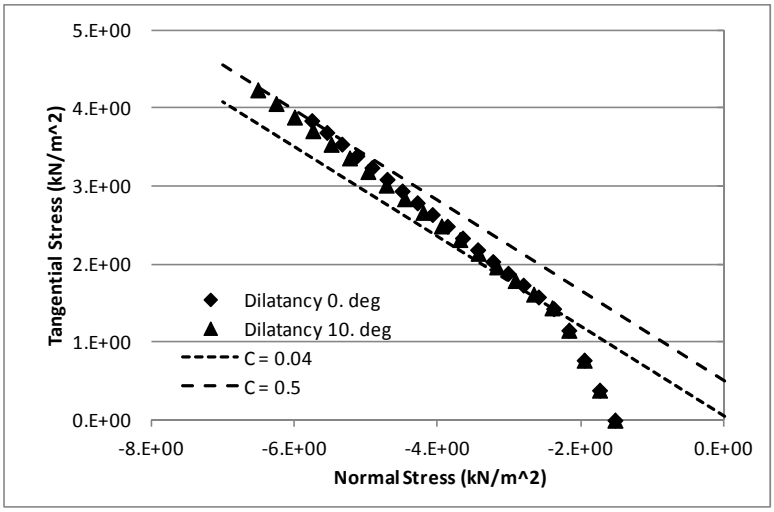


Figure 3.18.3
Unit test for interface elements with plastic dependent cohesion

In the above figure for plastic dependent cohesion, it can be shown that the cohesion grows as plastic relative displacement is increased and the guide line for C=0.5 exist only for the clear explanation purpose.

3.19 Unit Tests for Pile, Pile tip Elements

REFERENCE	Analytical solution
ELEMENTS	Pile , Pile tip elements
MODEL FILENAME	Material19_1.gts, Material19_2.gts

Unit tests are carried out for the nonlinear behavior of pile/pile tip elements.

Figure 3.19.1 depicts the test configuration. In the figure, the rectangular column is modeled with solid elements, and the beam inside the rectangular column is modeled with beam elements. Pile elements correlate two element types. Pile tip elements are attached at the end of the pile elements.

The nonlinear behaviors for pile, pile tip elements can be defined as perfect plastically using a ultimate force or to follow multi linear curve using table function. And if a table given, the stiffness for an element is determined according to the slope of the multi linear curve.
The table below show the material data for pile, pile tip elements and the thickness for pile elements.

The simulations are carried out with perfect plastic and a multi linear curve conditions.

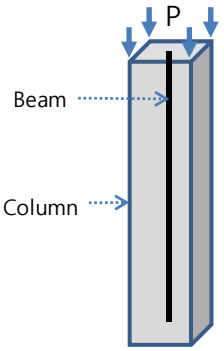


Figure 3.19.1.
Unit test configuration

All test results show that pile, pile tip elements exactly follow the given nonlinear behaviors. In the case of perfect plastic for pile elements, analytical solution is 0.15kN/m because of the pile thickness.

Material data for pile

Normal stiffness modulus	K_n = 80 kN/m^3
Shear stiffness modulus	K_t = 100 kN/m^3
Ultimate shear force	F_u = 1.5 kN/m/m
Normal stiffness modulus	K_n = 80 kN/m^3

Table

Displacement (m)	Force per unit thickness (kN/m/m)
-0.05	-2.0
-0.0187	-1.5
0.0	0.0
0.0187	1.5
0.05	2.0

Material data for pile tip	Tip spring stiffness	K = 100 kN/m
	Tip bearing capacity	F_u = 0.4 kN
	Table	

Displacement(m)	Force (kN)
-0.01	-0.5
-0.004	0.4
0.0	0.0
0.004	0.4
0.01	0.5

Pile property	thickness	0.1 m

For the following figure, probing points for pile element results are selected at the 2nd node of pile element at the top.

Figure 3.19.2
Unit test for pile elements

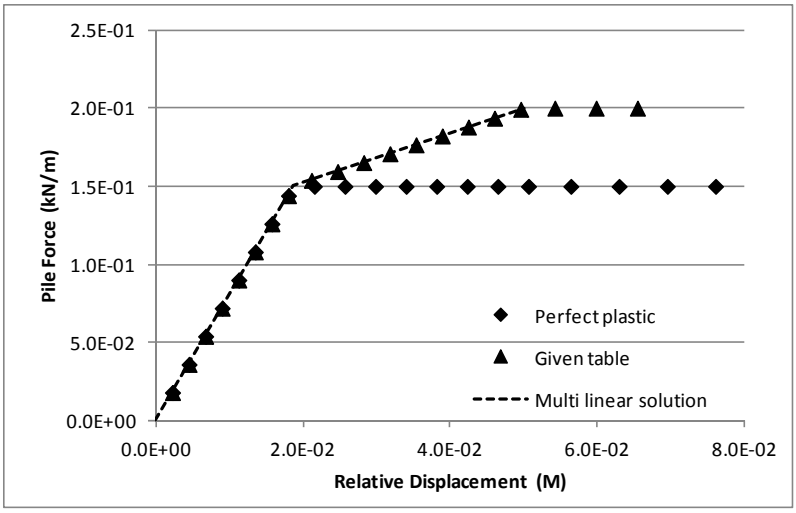
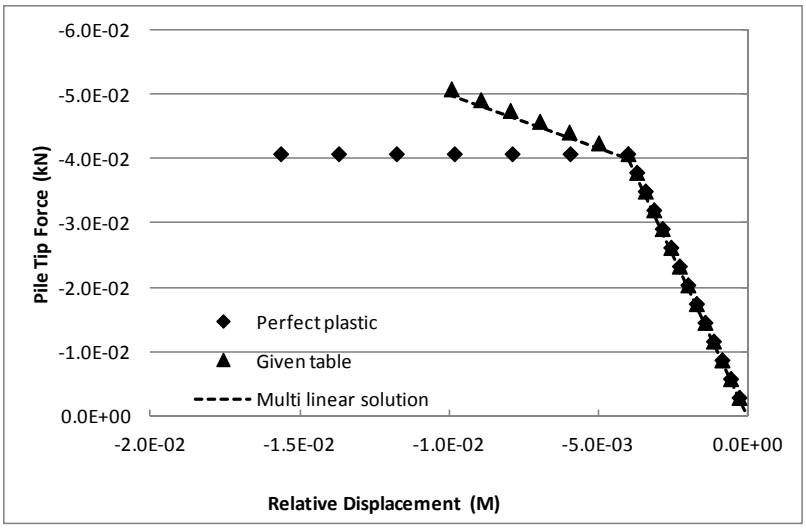


Figure 3.19.3
Unit test for pile tip elements



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